

Basics of Hybrid Simulation

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Outline

- 1. Hybrid Simulation (HS) Background**
- 2. HS Classification**
- 3. Benefits of HS**
- 4. Review of HS Related Research**



HS Background

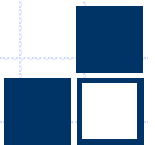
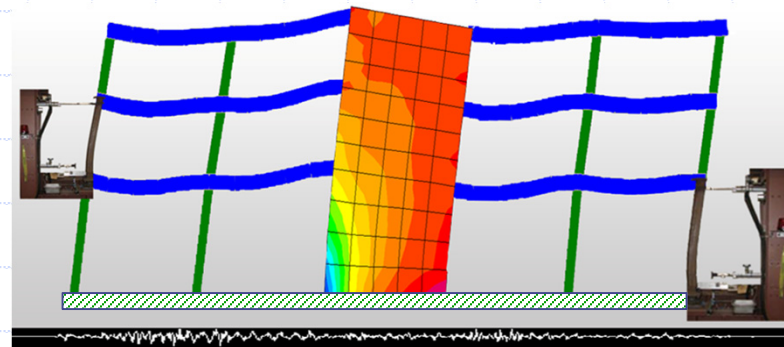
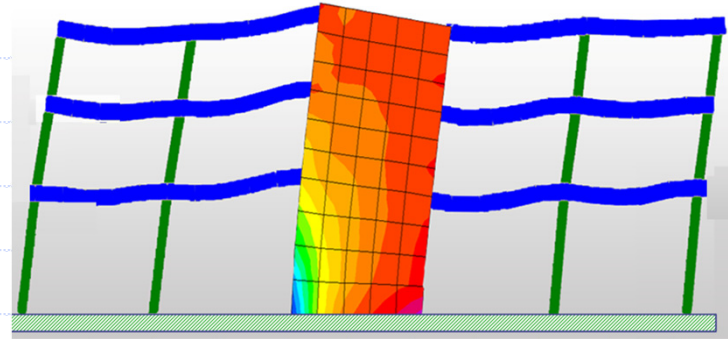
Analytical
Simulation

+

Experimental
Simulation

=

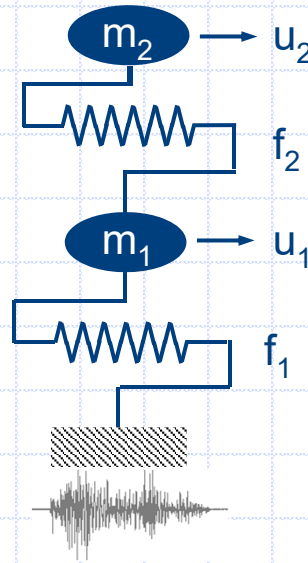
Hybrid
Simulation





HS Background

Analytical
Simulation:



$$-f_2 = m_2 \ddot{u}_2^t$$

$$-f_1 + f_2 = m_1 \ddot{u}_1^t$$

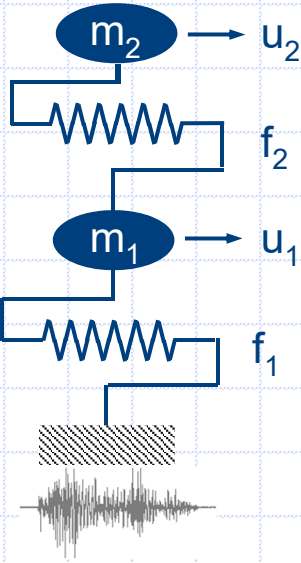
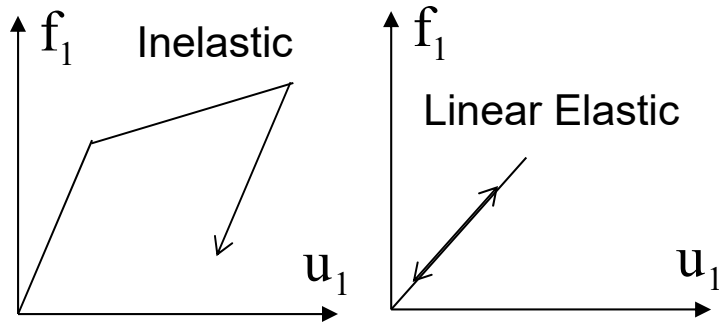
$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1^t \\ \ddot{u}_2^t \end{bmatrix} + \begin{bmatrix} -f_2 + f_1 \\ f_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\ddot{u}_1^t = \ddot{u}_1 + \ddot{u}_g \quad \ddot{u}_2^t = \ddot{u}_2 + \ddot{u}_g \quad \rightarrow \quad \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} -f_2 + f_1 \\ f_2 \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_g$$



HS Background

Analytical Simulation:



$$-f_2 = m_2 \ddot{u}_2^t$$

$$-f_1 + f_2 = m_1 \ddot{u}_1^t$$

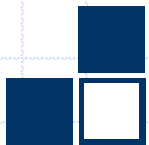
For the linear-elastic case

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} -f_2 + f_1 \\ f_2 \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_g$$

$$f_1 = k_1 u_1$$

$$f_2 = k_2 (u_2 - u_1)$$

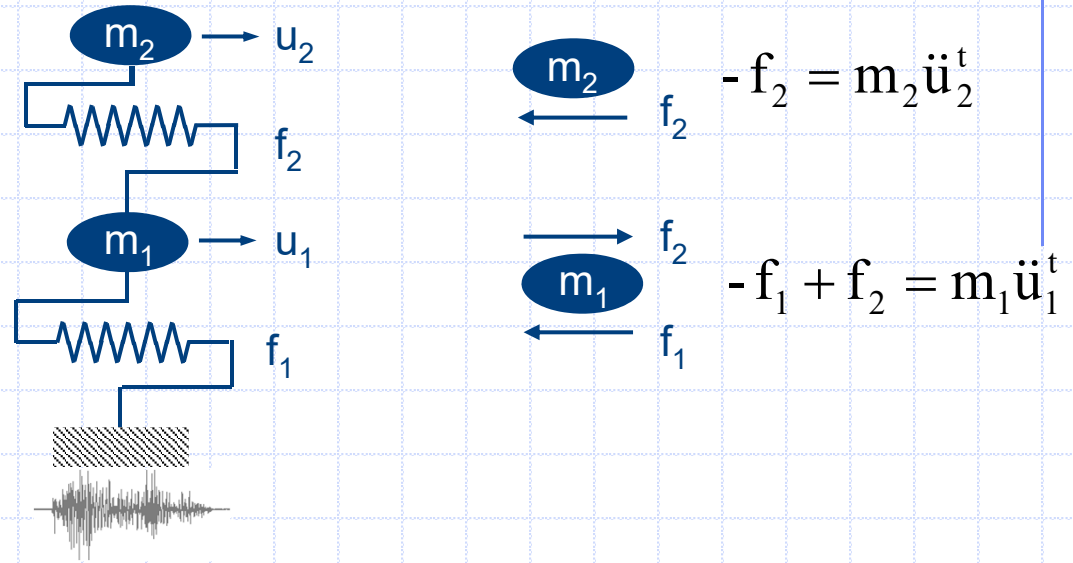
$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_g$$





HS Background

Analytical
Simulation:



$$C = \alpha M + \beta K$$

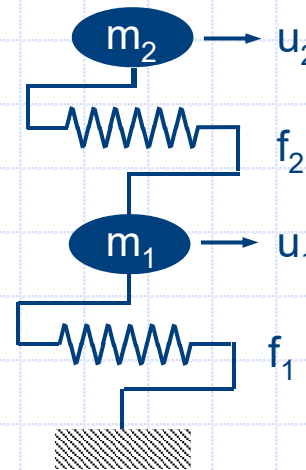
Damping is commonly represented as a linear combination of mass and stiffness matrices

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_g$$



HS Background

Analytical Simulation:



General case of inelastic response

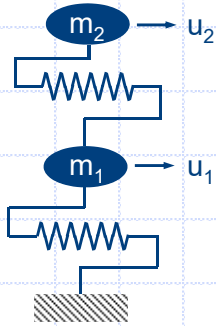
$$\underbrace{\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}}_{\mathbf{m}} \underbrace{\begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix}}_{\ddot{\mathbf{u}}} + \underbrace{\begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix}}_{\mathbf{c}} \underbrace{\begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \end{bmatrix}}_{\dot{\mathbf{u}}} + \underbrace{\begin{bmatrix} -f_2 + f_1 \\ f_2 \end{bmatrix}}_{\mathbf{f}} = \underbrace{\begin{bmatrix} m_1 \\ m_2 \end{bmatrix}}_{\mathbf{p}} \ddot{u}_g$$

$$\mathbf{m} \ddot{\mathbf{u}} + \mathbf{c} \dot{\mathbf{u}} + \mathbf{f} = \mathbf{p}$$

Objective of Analytical Simulation: Solve the equation of motion using numerical integration methods



HS Background



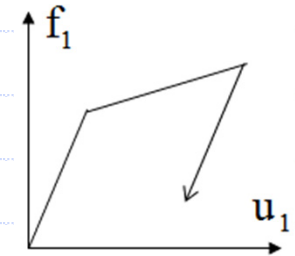
$$\underbrace{\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}}_{\mathbf{m}} \underbrace{\begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix}}_{\ddot{\mathbf{u}}} + \underbrace{\begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix}}_{\mathbf{c}} \underbrace{\begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \end{bmatrix}}_{\dot{\mathbf{u}}} + \underbrace{\begin{bmatrix} -f_2 + f_1 \\ f_2 \end{bmatrix}}_{\mathbf{f}} = - \underbrace{\begin{bmatrix} m_1 \\ m_2 \end{bmatrix}}_{\mathbf{p}} \ddot{u}_g$$

$$\mathbf{m} \ddot{\mathbf{u}} + \mathbf{c} \dot{\mathbf{u}} + \mathbf{f} = \mathbf{p}$$

A straightforward integration application: Explicit Newmark Integration

1) Compute the displacements $\mathbf{u}_{i+1} = \mathbf{u}_i + \Delta t \dot{\mathbf{u}}_i + \frac{(\Delta t)^2}{2} \ddot{\mathbf{u}}_i$

2) Compute the restoring forces \mathbf{f}_{i+1} corresponding to \mathbf{u}_{i+1}



3) Compute the accelerations $[\mathbf{m} + \Delta t \gamma \mathbf{c}] \ddot{\mathbf{u}}_{i+1} = \mathbf{p}_{i+1} - \mathbf{f}_{i+1} - \mathbf{c} [\dot{\mathbf{u}}_i + \Delta t (1 - \gamma) \ddot{\mathbf{u}}_i]$

$$\mathbf{m}_{\text{eff}} \ddot{\mathbf{u}}_{i+1} = \mathbf{p}_{\text{eff}}$$

4) Compute the velocities $\dot{\mathbf{u}}_{i+1} = \dot{\mathbf{u}}_i + \Delta t [(1 - \gamma) \ddot{\mathbf{u}}_i + \gamma \ddot{\mathbf{u}}_{i+1}]$

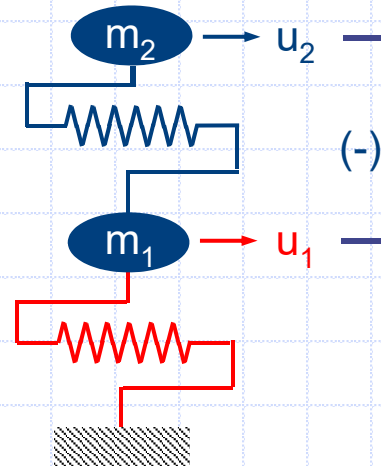
5) Increment i



HS Background

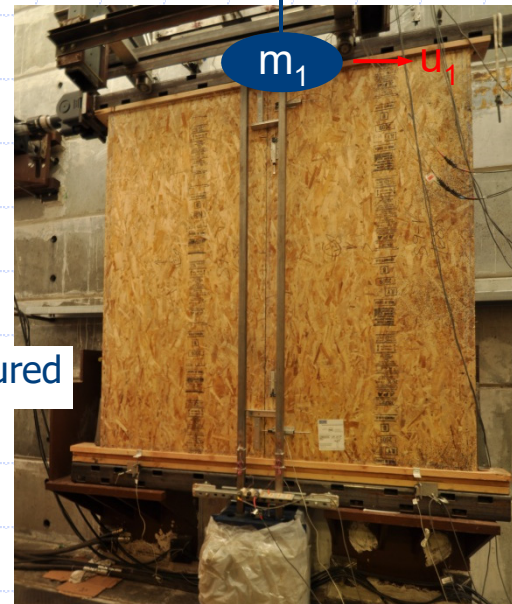


Bottom spring replaced with a test specimen



Measured

Computed



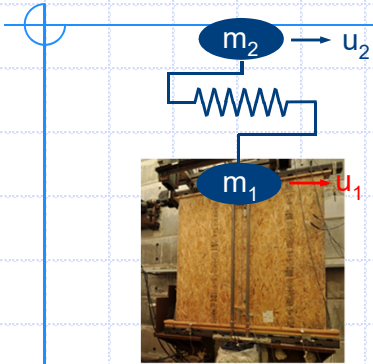
Analytical substructure

Experimental substructure

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \end{bmatrix} + \begin{bmatrix} -f_a + f_e \\ f_a \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_{gg}$$



HS Background



$$\underbrace{\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}}_{\mathbf{m}} \underbrace{\begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix}}_{\ddot{\mathbf{u}}} + \underbrace{\begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix}}_{\mathbf{c}} \underbrace{\begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \end{bmatrix}}_{\dot{\mathbf{u}}} + \underbrace{\begin{bmatrix} -f_a & -f_e \\ f_a \end{bmatrix}}_{\mathbf{f}} = \underbrace{\begin{bmatrix} m_1 \\ m_2 \end{bmatrix}}_{\mathbf{p}} \ddot{\mathbf{u}}_g$$

$$\mathbf{m} \ddot{\mathbf{u}} + \mathbf{c} \dot{\mathbf{u}} + \mathbf{f} = \mathbf{p}$$

A straightforward integration application: Explicit Newmark Integration

- 1) Compute the displacements $\mathbf{u}_{i+1} = \mathbf{u}_i + \Delta t \dot{\mathbf{u}}_i + \frac{(\Delta t)^2}{2} \ddot{\mathbf{u}}_i$
- 2a) Compute the restoring force $f_{a,i+1}$ corresponding to the displacement $u_{2,i+1} - u_{1,i+1}$
- 2b) Impose $u_{1,i+1}$ to the test specimen and measure the corresponding force $f_{e,i+1}$

- 3) Compute the accelerations $[\mathbf{m} + \Delta t \gamma \mathbf{c}] \ddot{\mathbf{u}}_{i+1} = \mathbf{p}_{i+1} - \mathbf{f}_{i+1} - \mathbf{c} [\dot{\mathbf{u}}_i + \Delta t (1 - \gamma) \ddot{\mathbf{u}}_i]$

$$\mathbf{m}_{\text{eff}} \ddot{\mathbf{u}}_{i+1} = \mathbf{p}_{\text{eff}}$$

- 4) Compute the velocities $\dot{\mathbf{u}}_{i+1} = \dot{\mathbf{u}}_i + \Delta t [(1 - \gamma) \ddot{\mathbf{u}}_i + \gamma \ddot{\mathbf{u}}_{i+1}]$

- 5) Increment i



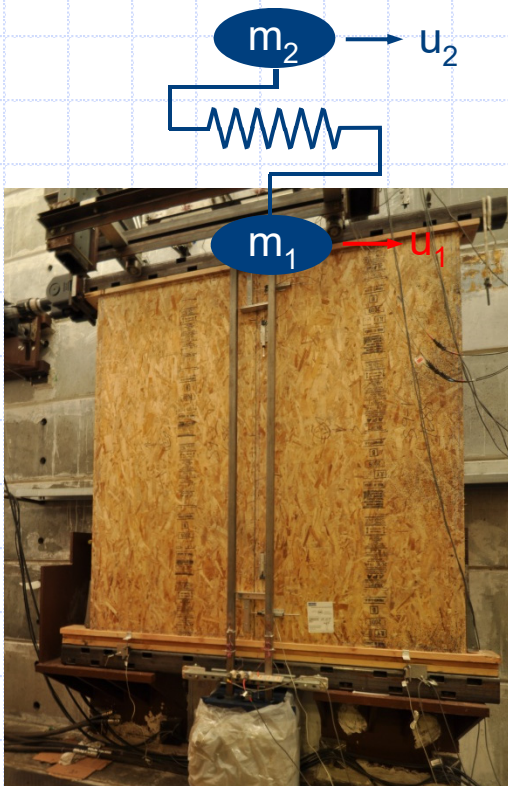
HS Classification

- ❑ Slow Hybrid Simulation
- ❑ Real-time Hybrid Simulation
 - Actuator Configuration
 - Shaking Table Configuration
 - Actuator + Shaking Table Configuration



HS Classification

❑ Slow Hybrid Simulation



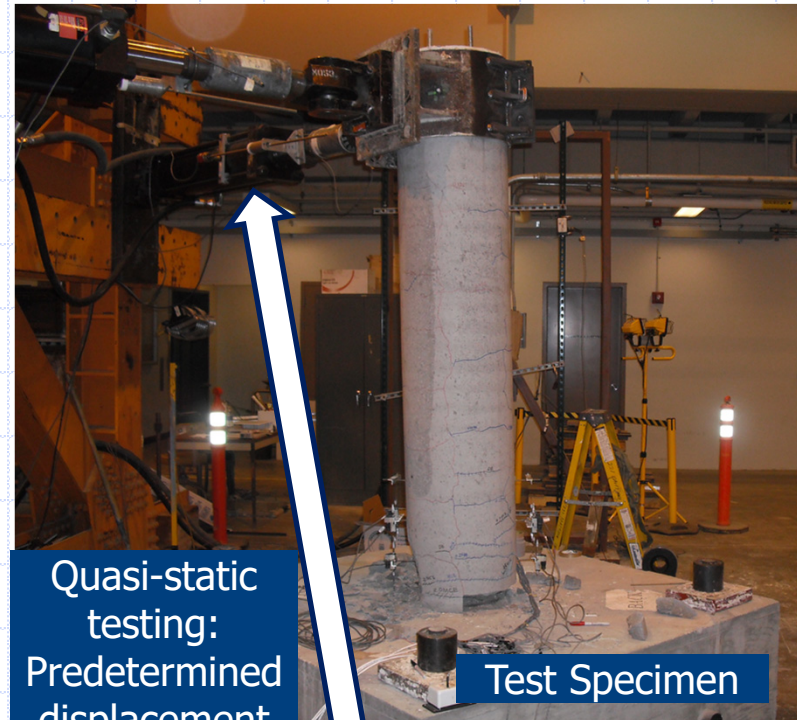
- ✓ Rate of loading $<$ Computed velocity
- ✓ Duration of hybrid simulation $>$ $N \times \Delta t$
N: number of integration steps
 Δt : integration time step
- ✓ Applicable when rate effects are not important
- ✓ Experimental substructure is connected to actuator(s)
- ✓ Physical mass generally doesn't exist



HS Classification

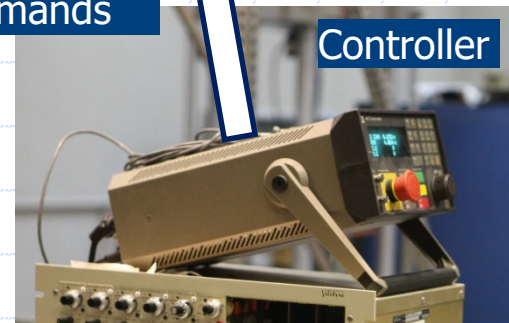
From the experimental perspective, slow hybrid simulation is equivalent to quasi-static testing

Predetermined displacement commands are based on a load protocol



Quasi-static testing:
Predetermined displacement commands

Test Specimen

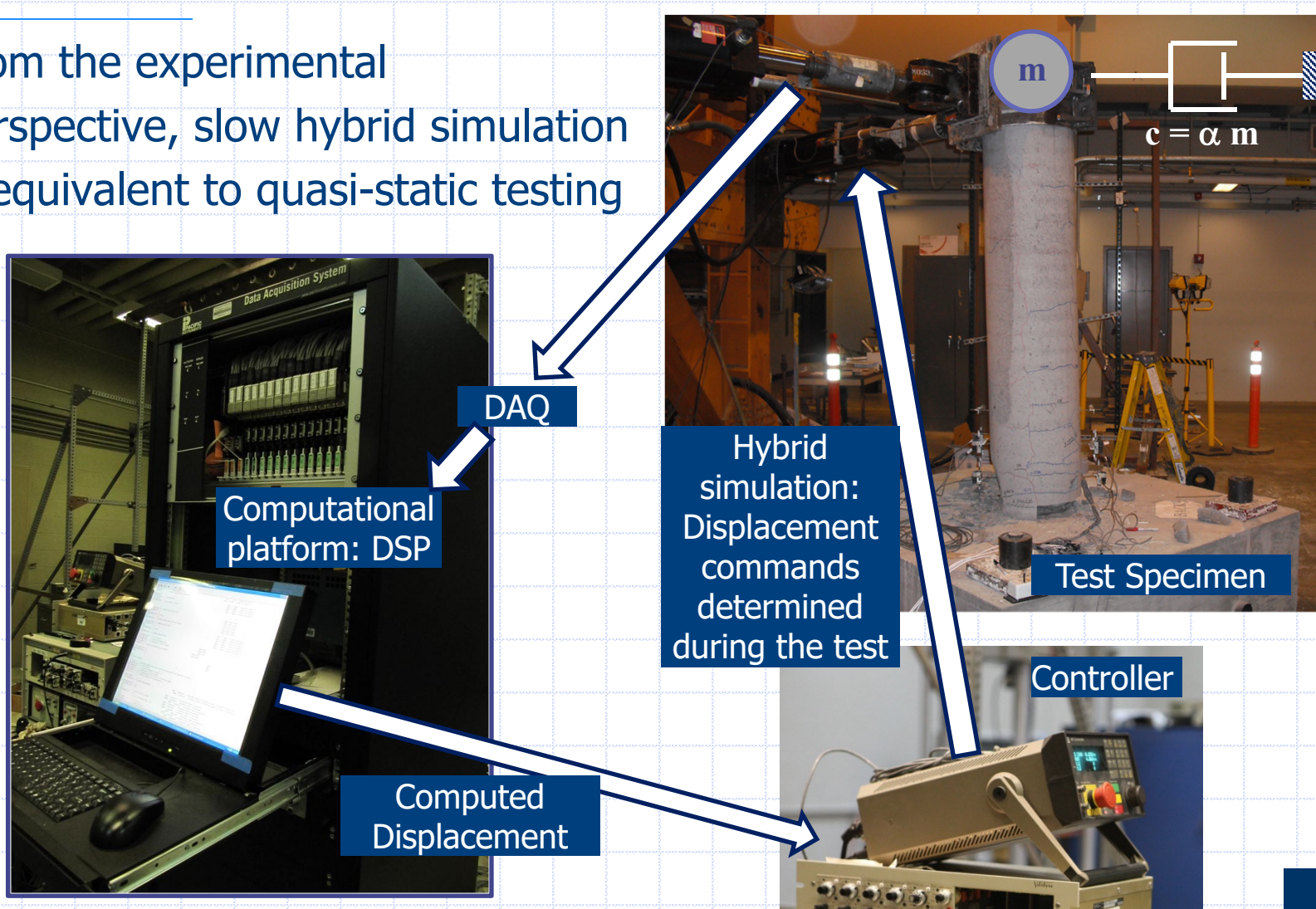


Controller



HS Classification

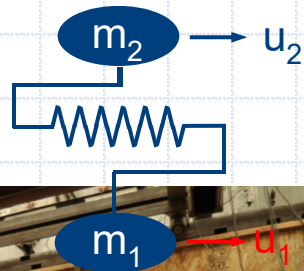
From the experimental perspective, slow hybrid simulation is equivalent to quasi-static testing





HS Classification

□ Real-time Hybrid Simulation (Actuator Configuration)

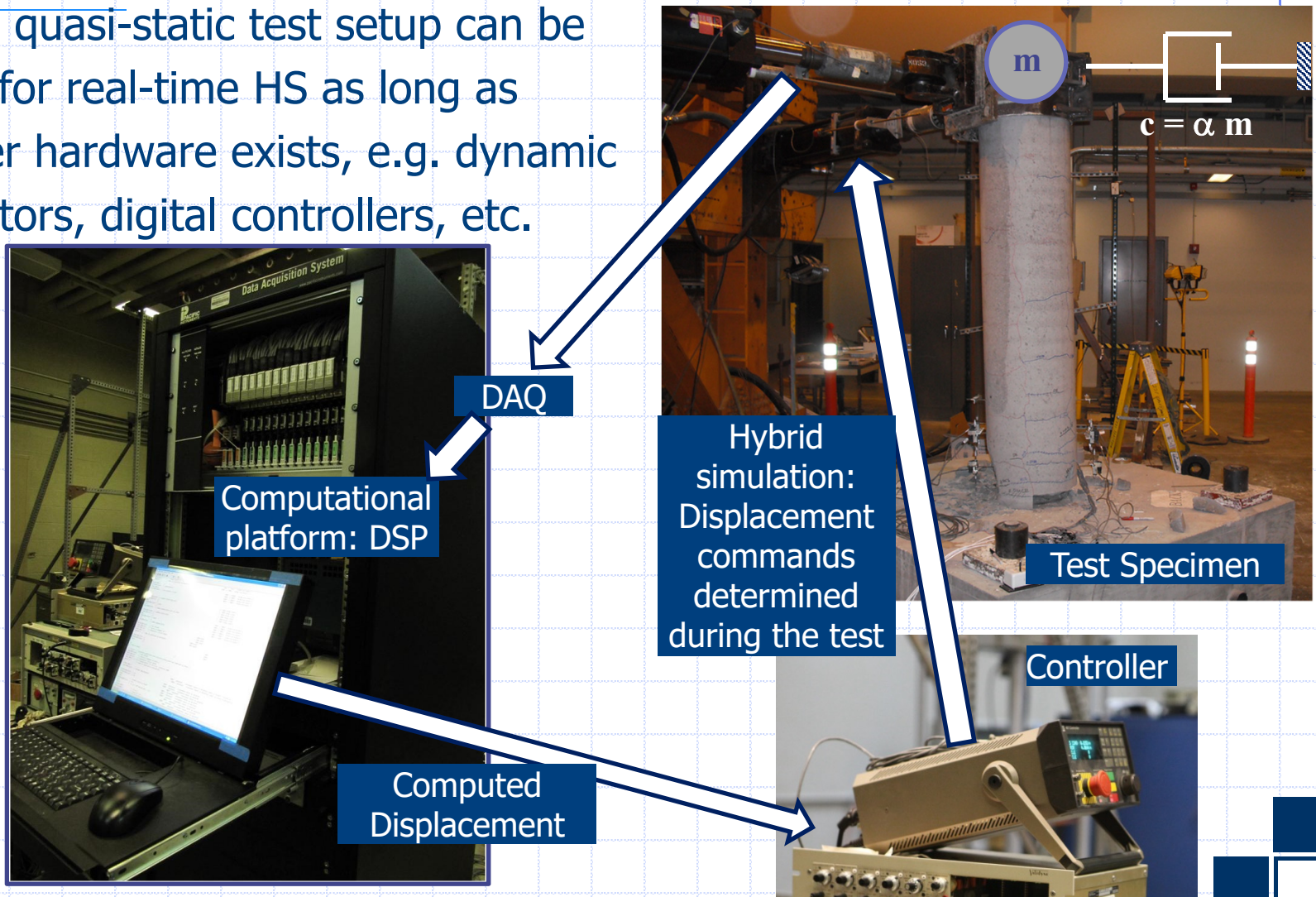


- ✓ Rate of loading = Computed velocity
- ✓ Duration of hybrid simulation = $N \times \Delta t$
N: number of integration steps
 Δt : integration time step
- ✓ Crucial when rate effects are important
- ✓ Experimental substructure is connected to actuator(s)
- ✓ Physical mass generally doesn't exist



HS Classification

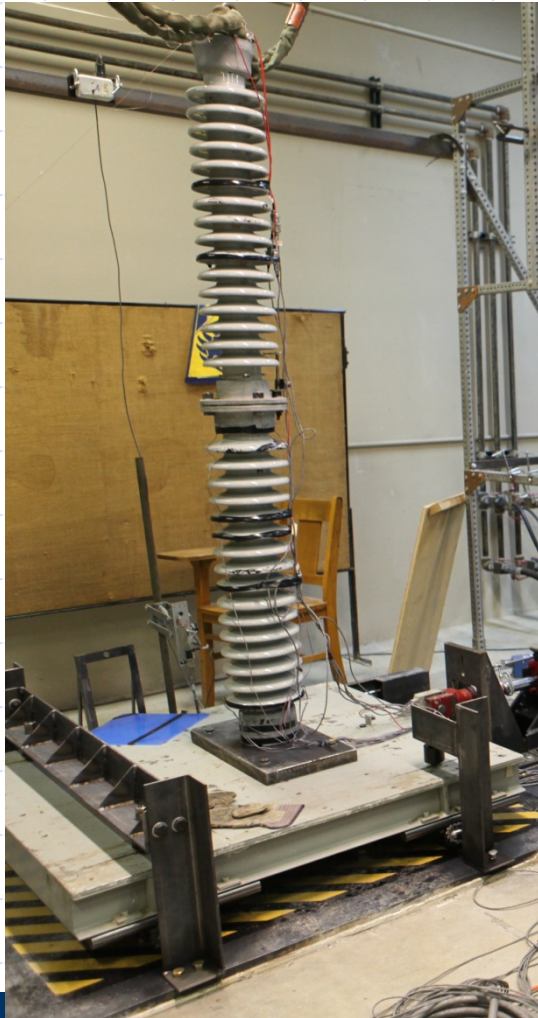
Same quasi-static test setup can be used for real-time HS as long as proper hardware exists, e.g. dynamic actuators, digital controllers, etc.





HS Classification

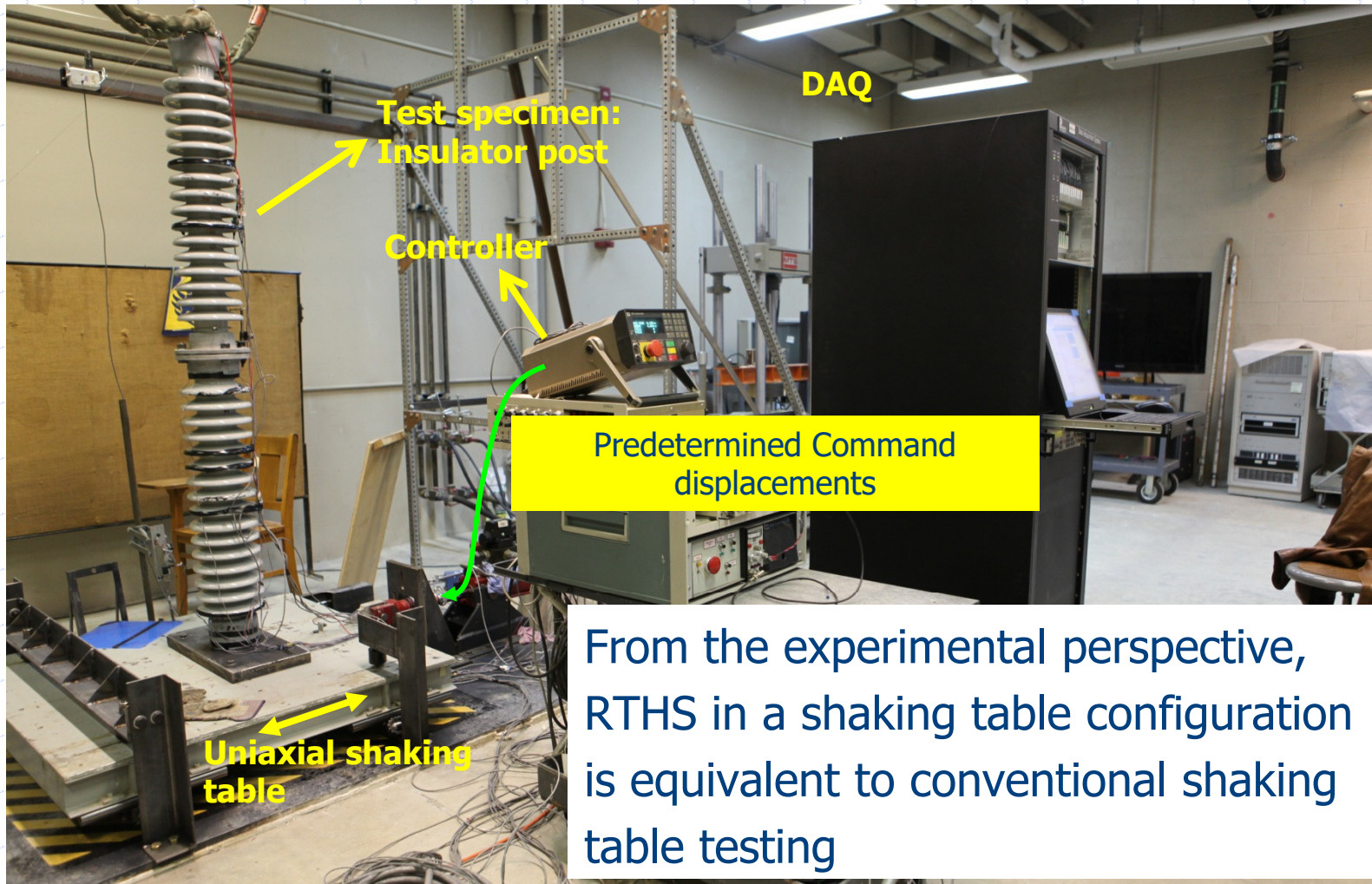
❑ Real-time Hybrid Simulation (Shaking Table Configuration)



- ✓ Experimental substructure is located on a shaking table
- ✓ Physical mass generally exists
- ✓ Rate of loading = Computed velocity
- ✓ Duration of hybrid simulation = $N \times \Delta t$
N: number of integration steps
 Δt : integration time step
- ✓ Crucial when rate effects are important

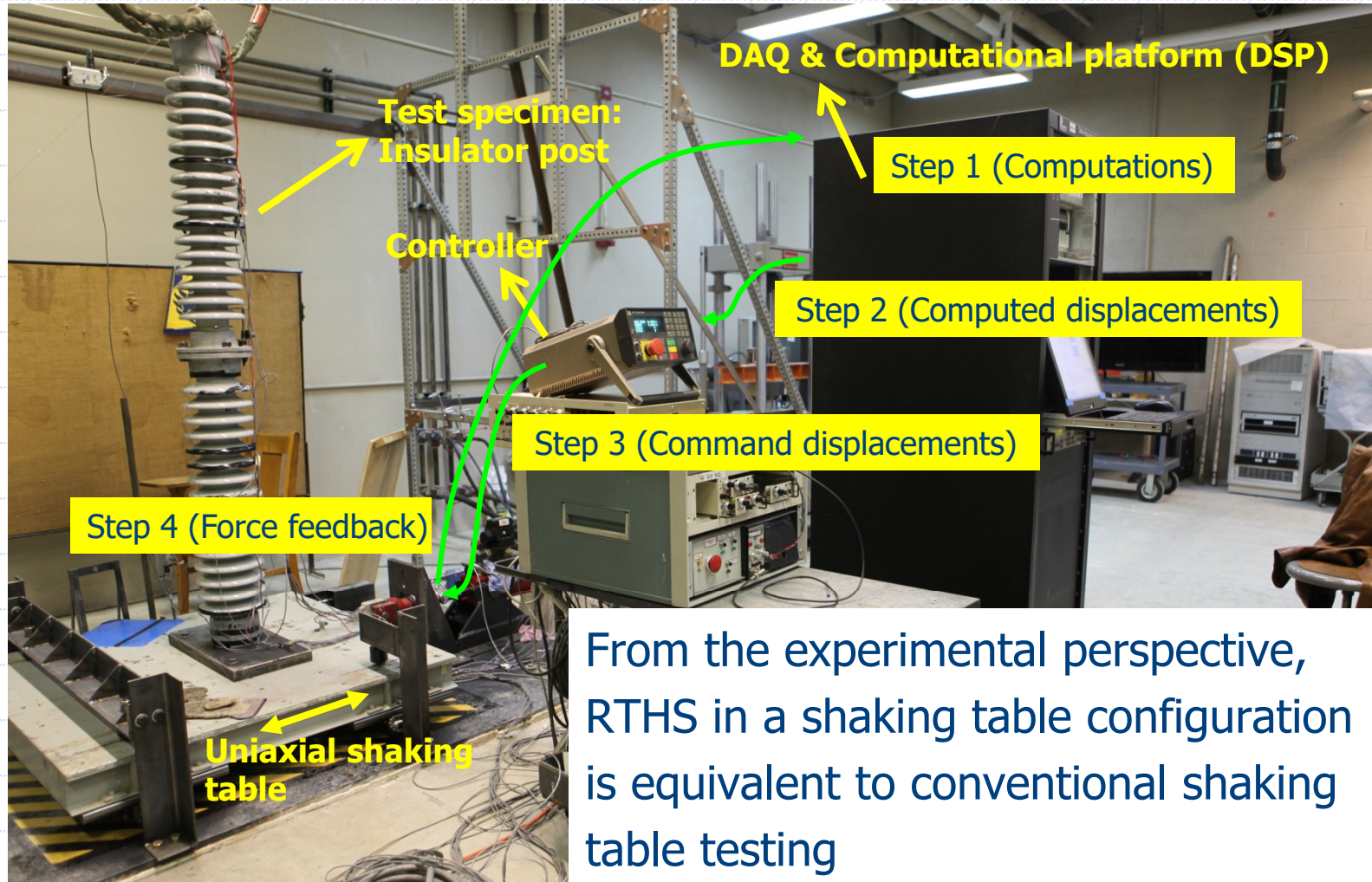


HS Classification





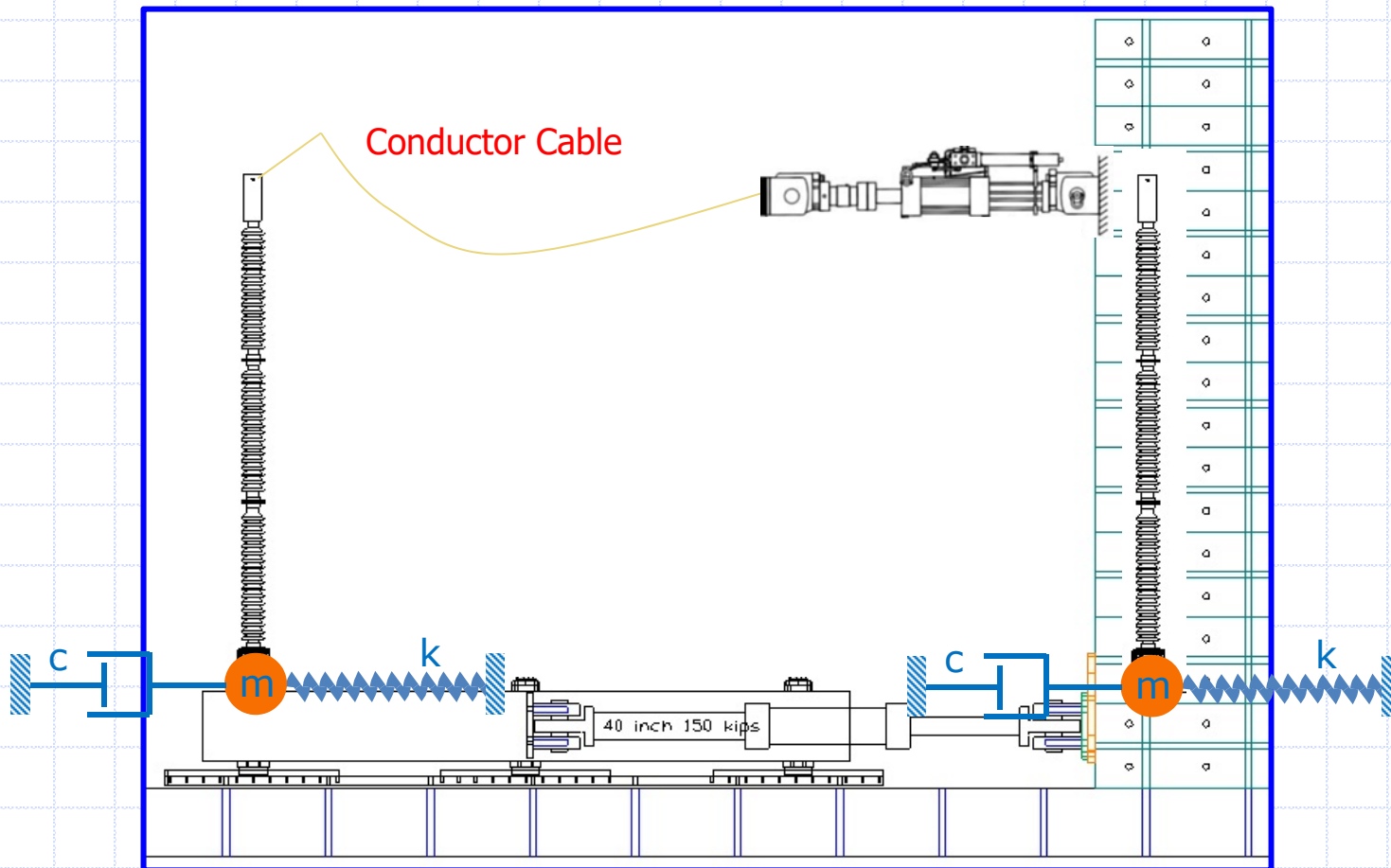
HS Classification





HS Classification

- Real-time Hybrid Simulation (Actuator + Shaking Table Configuration)



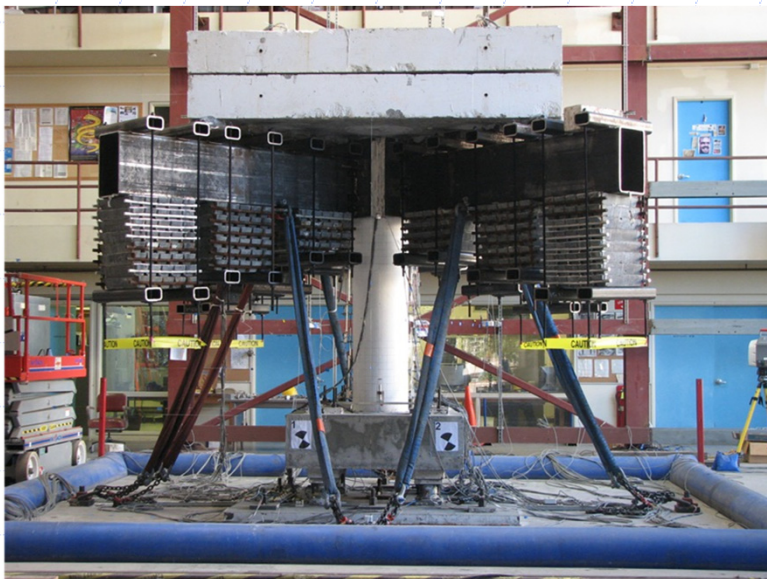
- Experimental substructure is located on a shaking table and connected to an actuator



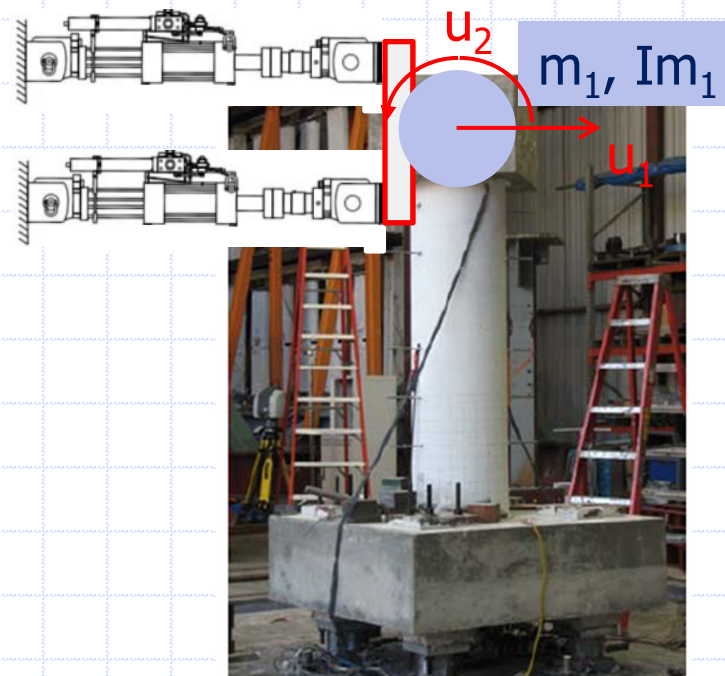
Benefits of HS

Convenience in mass modeling

Shaking Table



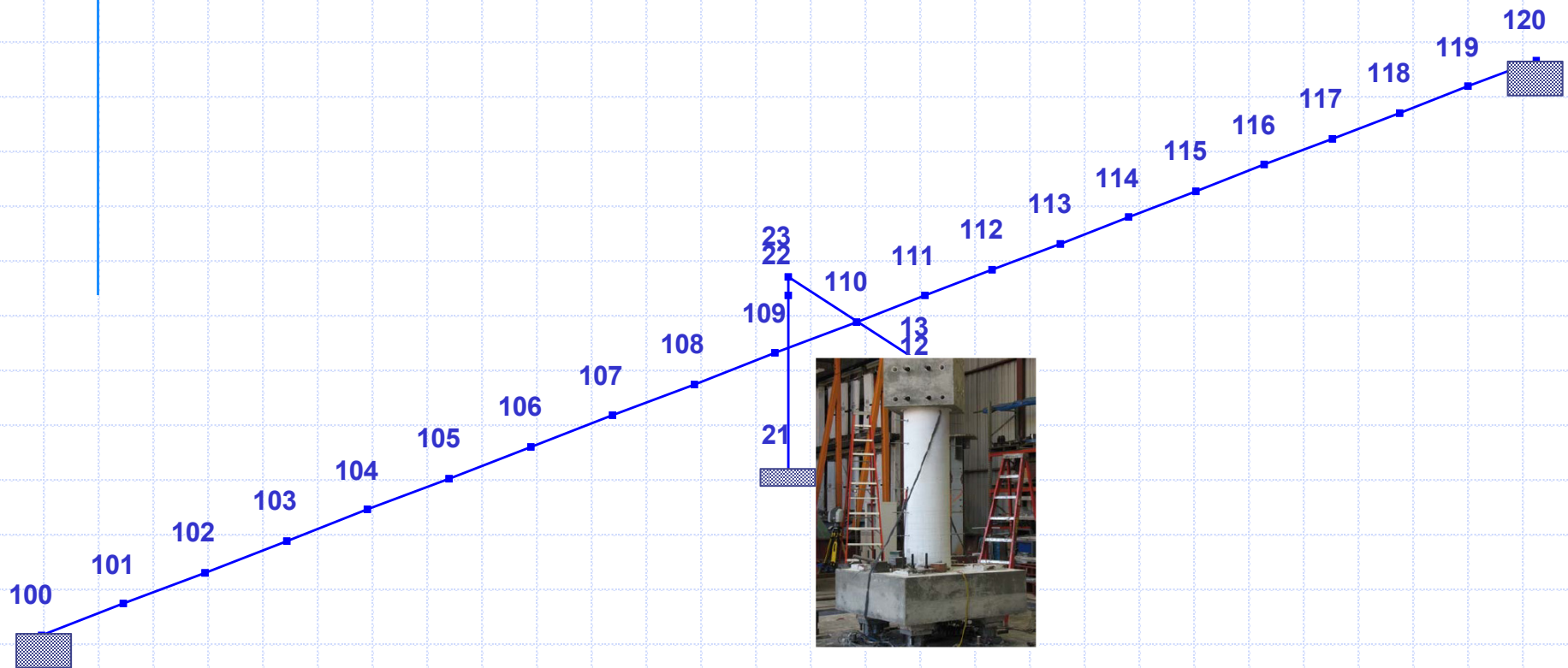
Hybrid Simulation





Benefits of HS

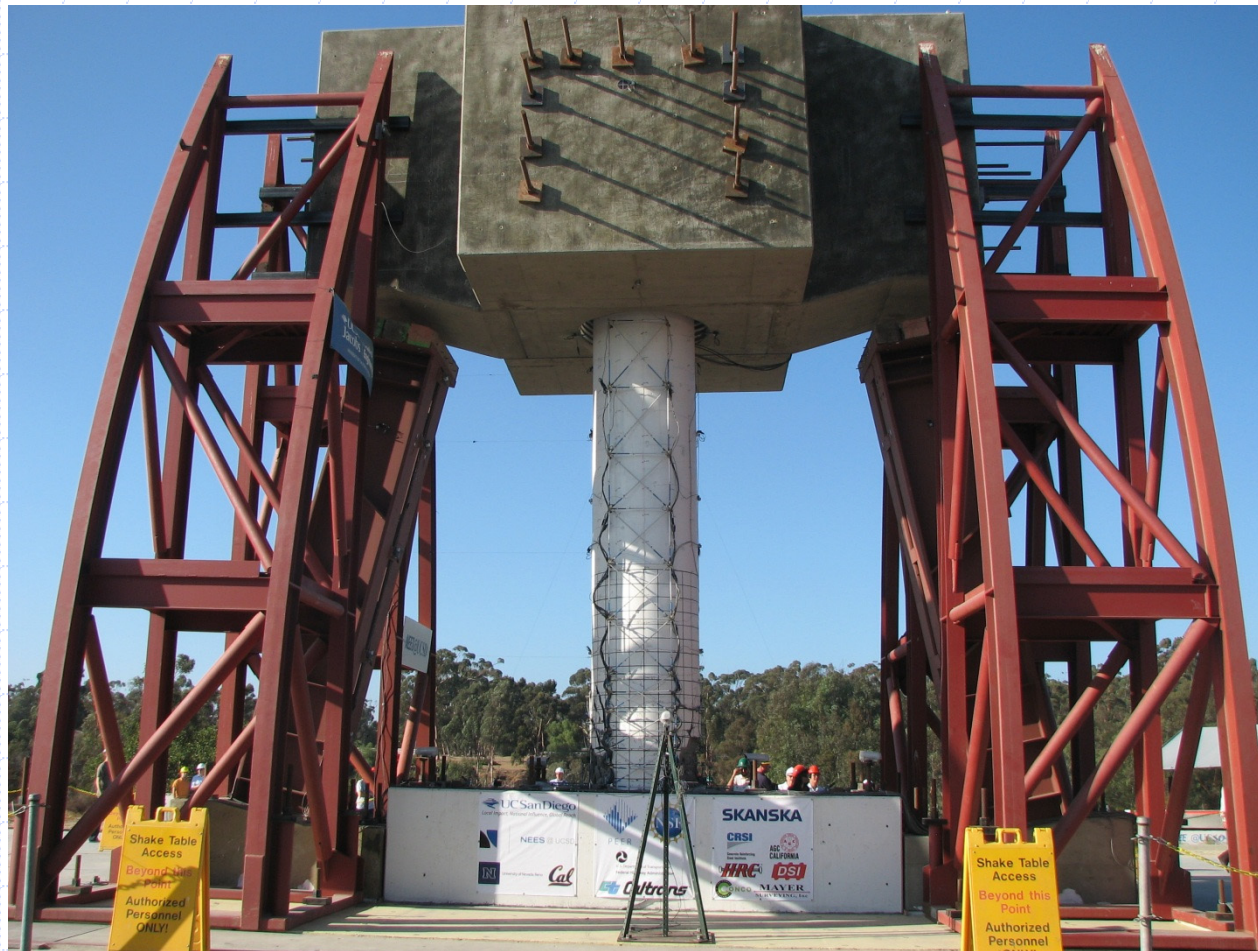
Convenience in system level testing





Benefits of HS

Convenience in mass modeling





Benefits of HS

Convenience in full scale testing



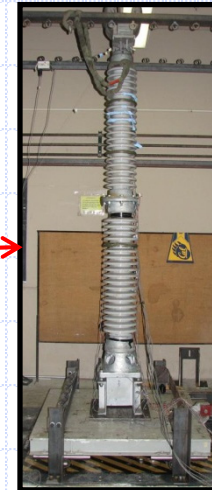


Benefits of HS

Time efficiency due to elimination of physical construction



Experimental substructure

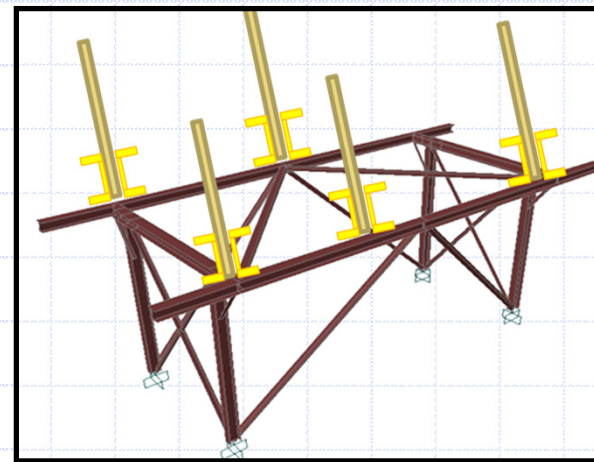


Details in the HS application lecture

≡

+

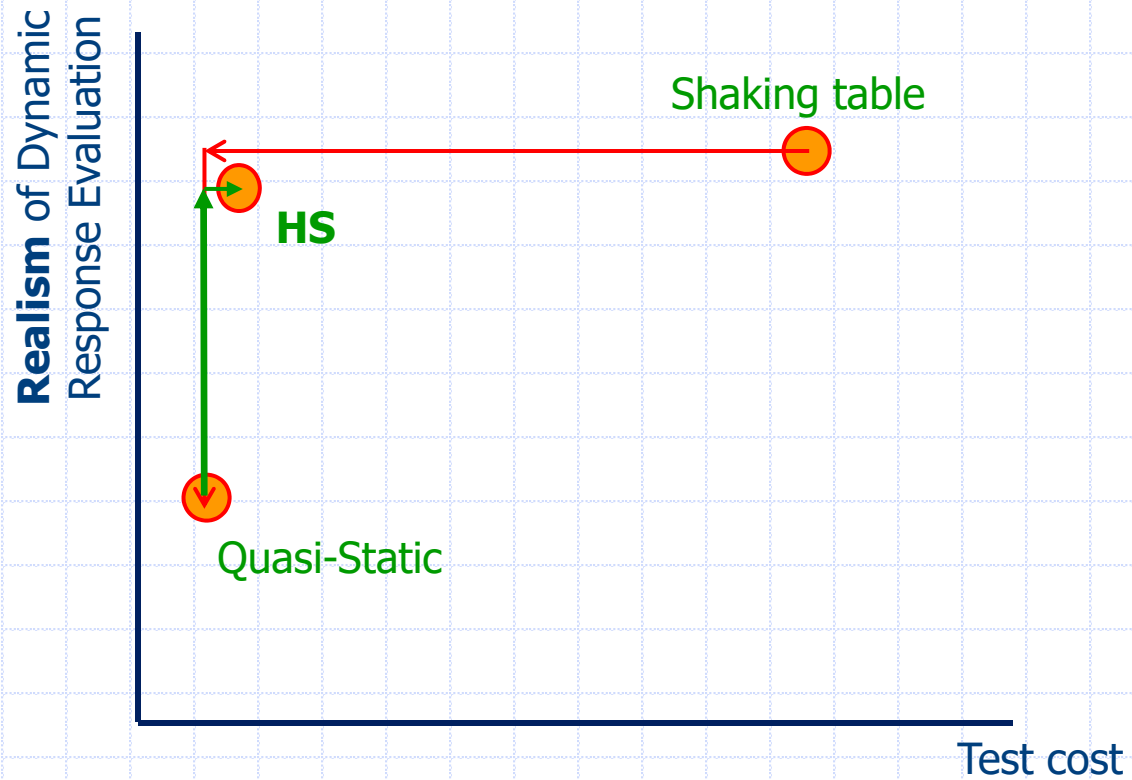
Analytical substructure





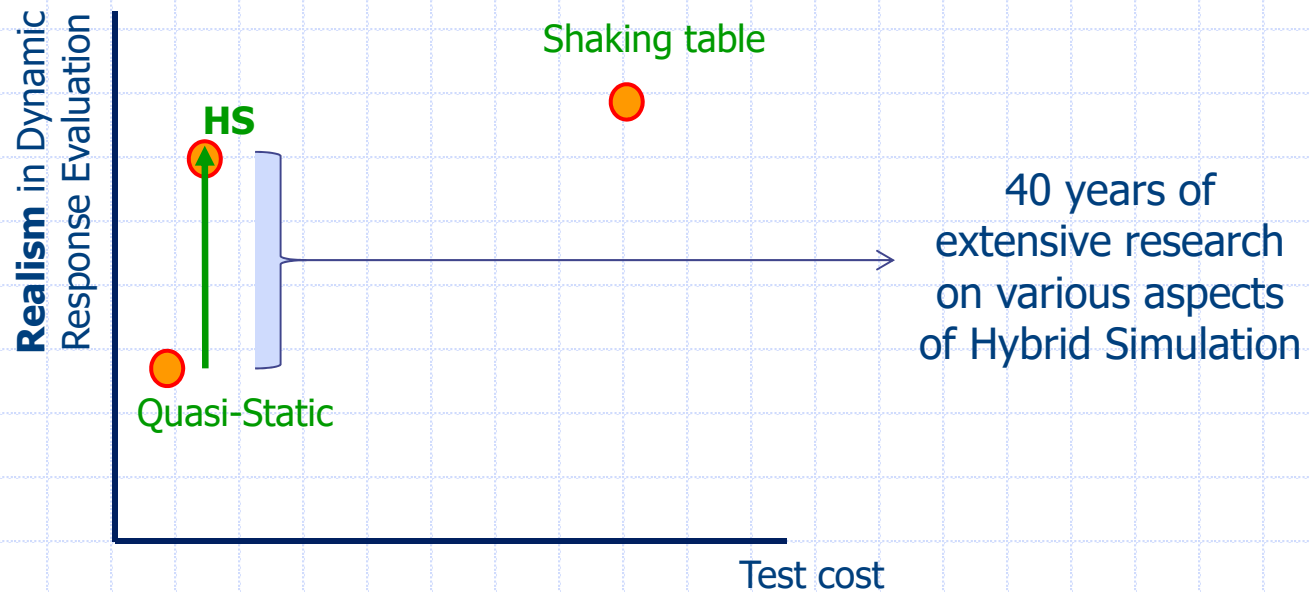
Benefits of HS

Economical Convenience





Benefits of HS

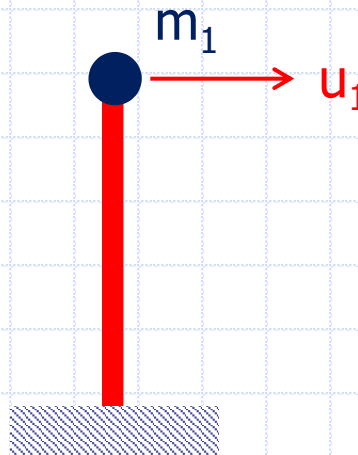


- Nature of the problem requires **substructuring**
- Presence of experimental substructures require the use of special **integration methods**
- Presence of a transfer system introduce **simulation errors**
- Rate dependent materials require **real-time hybrid simulation (RTHS)**
- Making use of multiple labs extend the method to **geographically distributed testing**

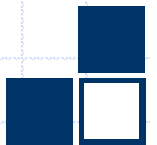


Substructuring Cases

CASE 1: CANTILEVER COLUMN with MASS [No MASS MOMENT of INERTIA or ANALYTICAL SUBSTRUCTURE]



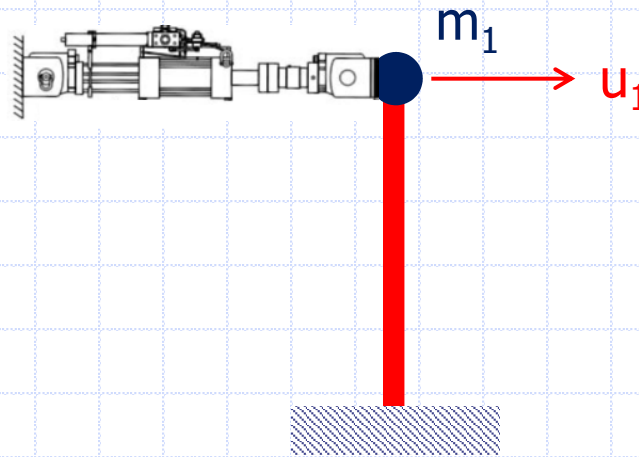
Red : Experimental
Blue: Analytical



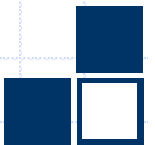


Substructuring Cases

CASE 1: CANTILEVER COLUMN with MASS [No MASS MOMENT of INERTIA or ANALYTICAL SUBSTRUCTURE]



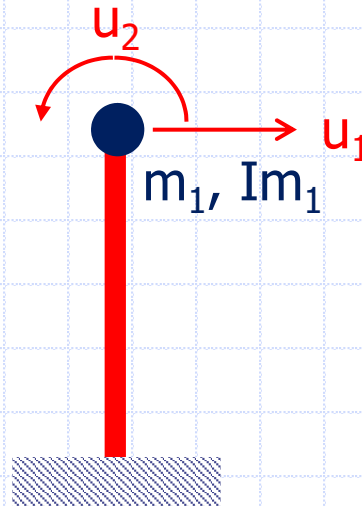
Red : Experimental
Blue: Analytical





Substructuring Cases

CASE 2: CANTILEVER COLUMN with MASS and MASS MOMENT of INERTIA [No ANALYTICAL SUBSTRUCTURE]



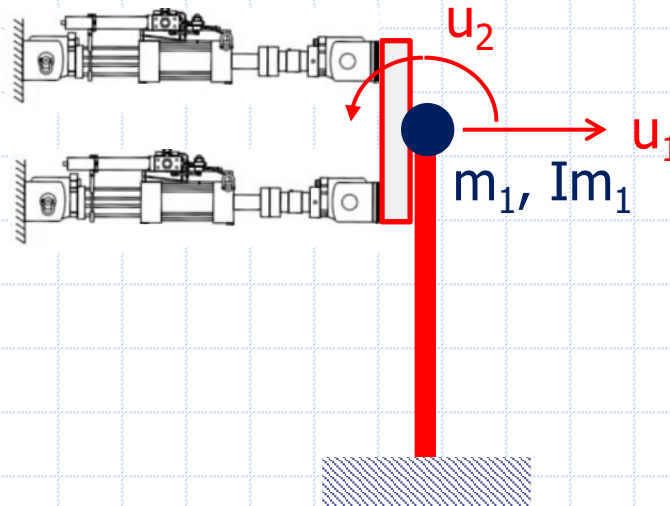
Red : Experimental
Blue: Analytical



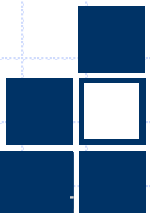


Substructuring Cases

CASE 2: CANTILEVER COLUMN with MASS and MASS MOMENT of INERTIA [No ANALYTICAL SUBSTRUCTURE]



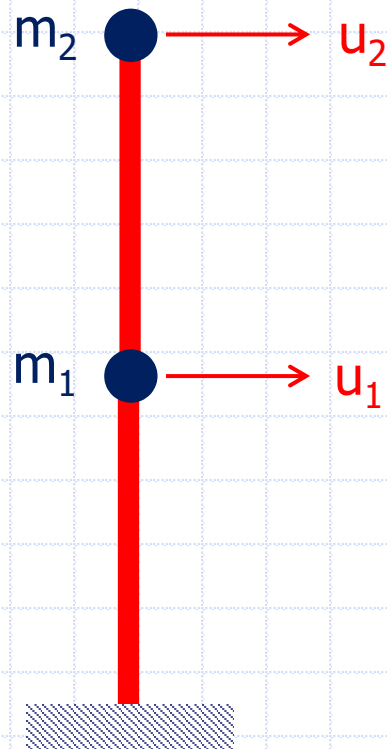
Red : Experimental
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Substructuring Cases

CASE 3: TWO COLUMNS without ANALYTICAL SUBSTRUCTURE



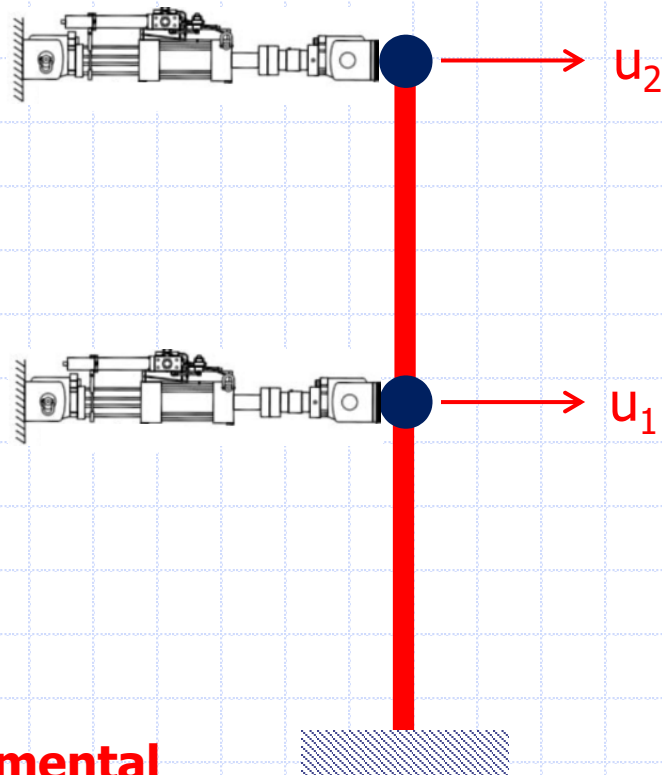
Red : Experimental
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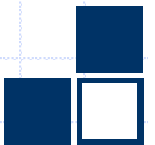


Substructuring Cases

CASE 3: TWO COLUMNS without ANALYTICAL SUBSTRUCTURE



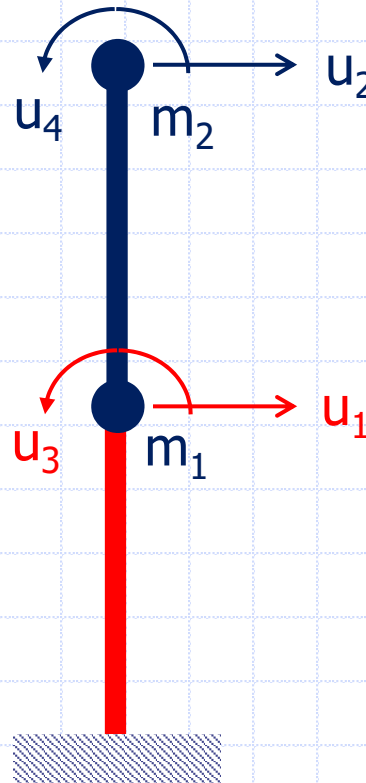
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Blue: Analytical



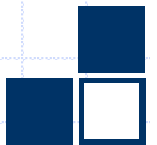


Substructuring Cases

CASE 4: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



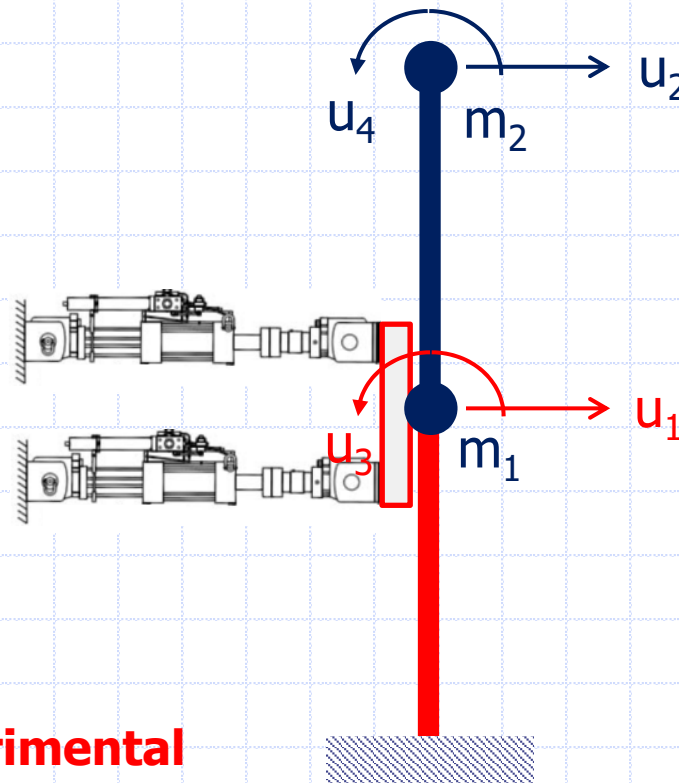
Red : Experimental
Blue: Analytical



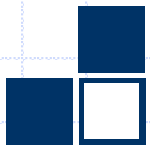


Substructuring Cases

CASE 4: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



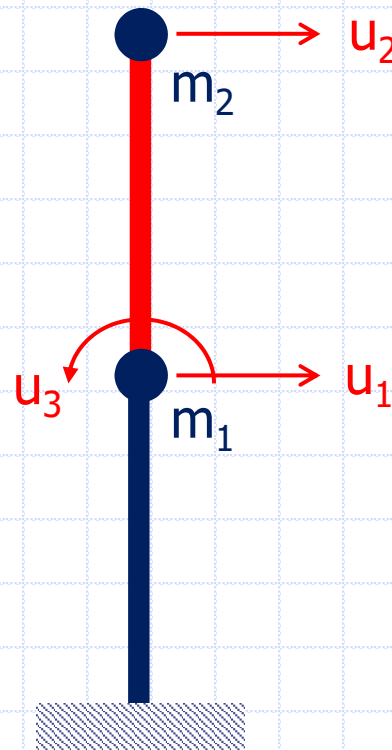
Red : Experimental
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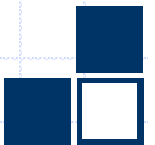


Substructuring Cases

CASE 4-1: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



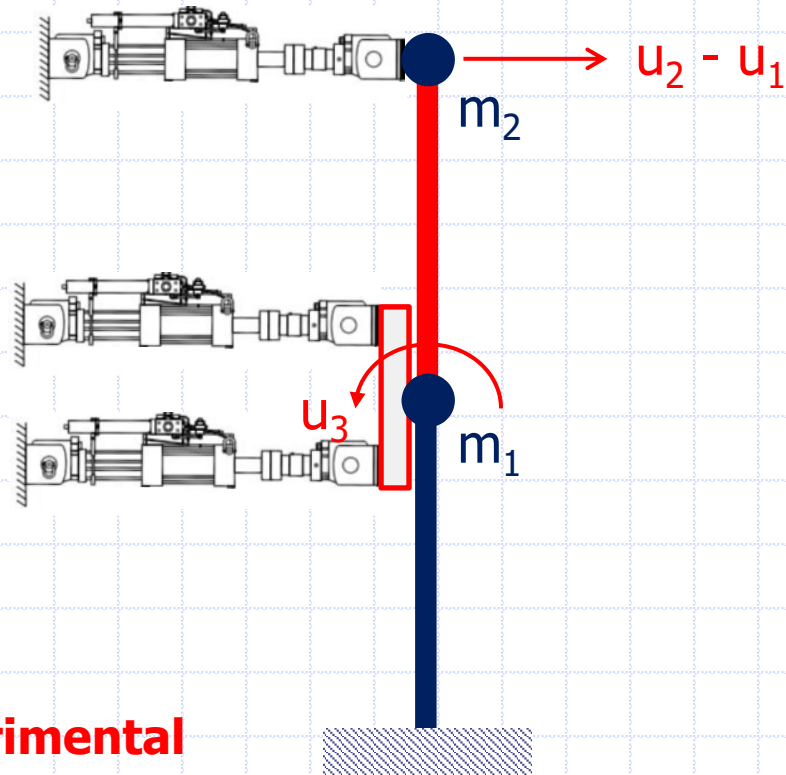
Red : Experimental
Blue: Analytical



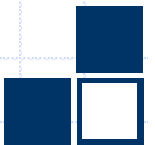


Substructuring Cases

CASE 4-1: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



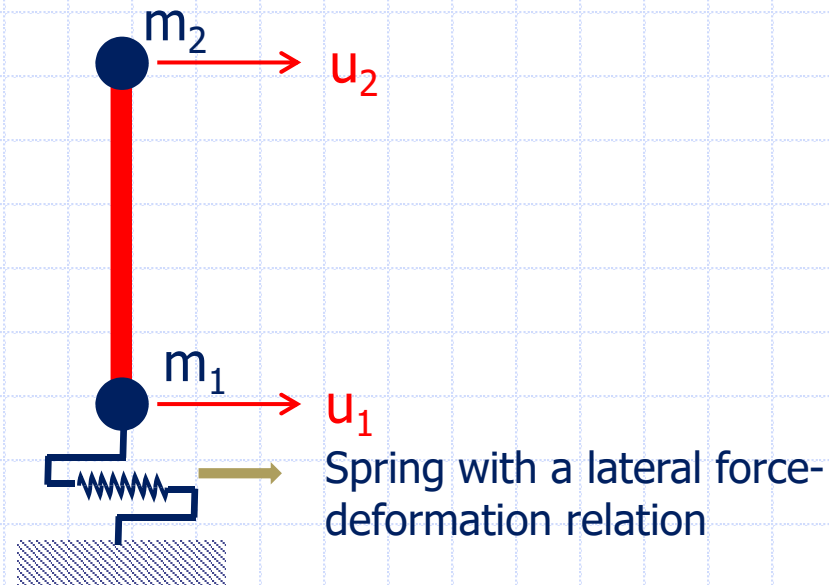
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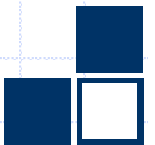


Substructuring Cases

CASE 4-2: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



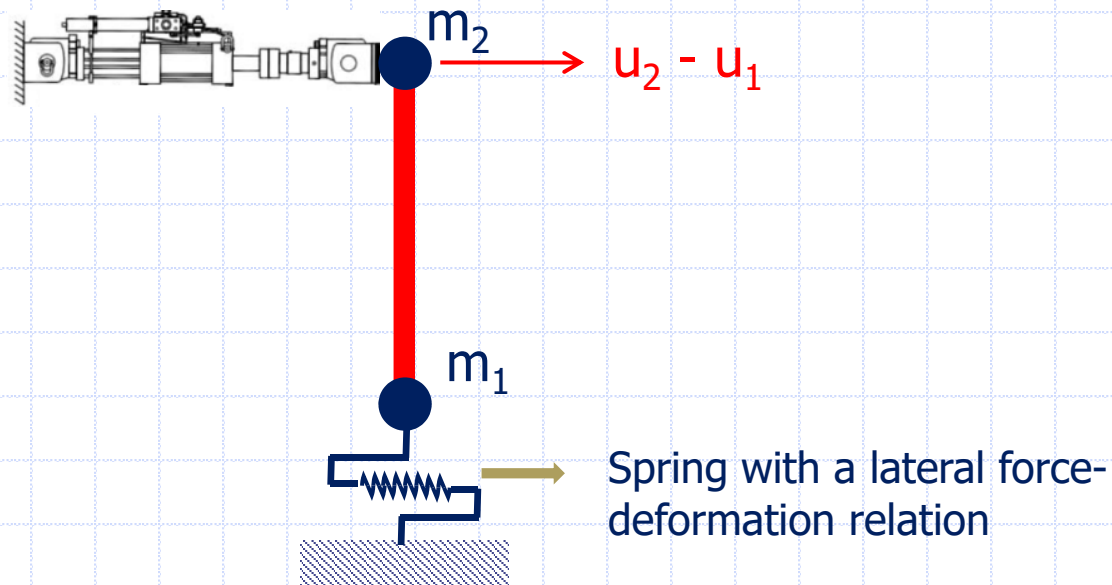
Red : Experimental
Blue: Analytical



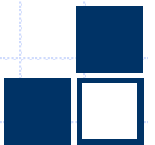


Substructuring Cases

CASE 4-2: TWO COLUMNS with an EXPERIMENTAL and an ANALYTICAL SUBSTRUCTURE



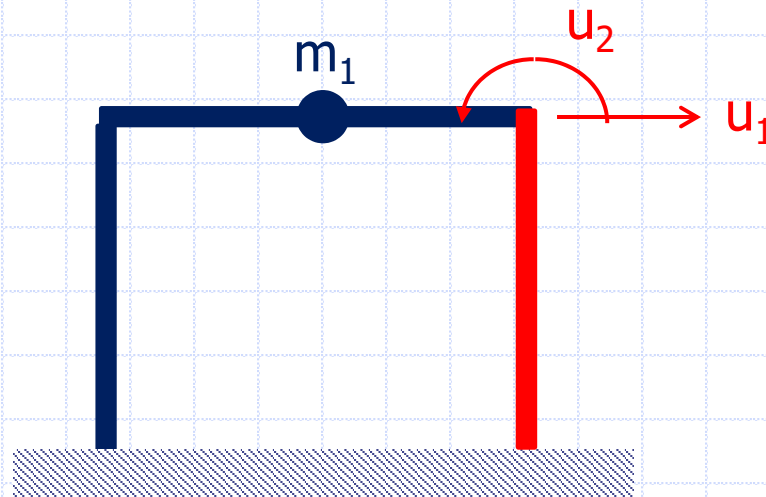
Red : Experimental
Blue: Analytical





Substructuring Cases

CASE 5: PORTAL FRAME with ONE OF THE COLUMNS AND BEAM AS ANALYTICAL SUBSTRUCTURE



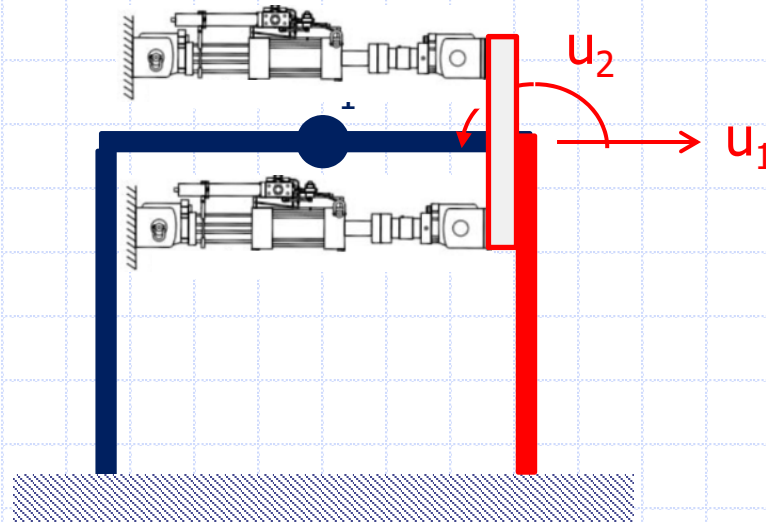
Red : Experimental
Blue: Analytical





Substructuring Cases

CASE 5: PORTAL FRAME with ONE OF THE COLUMNS AND BEAM AS ANALYTICAL SUBSTRUCTURE



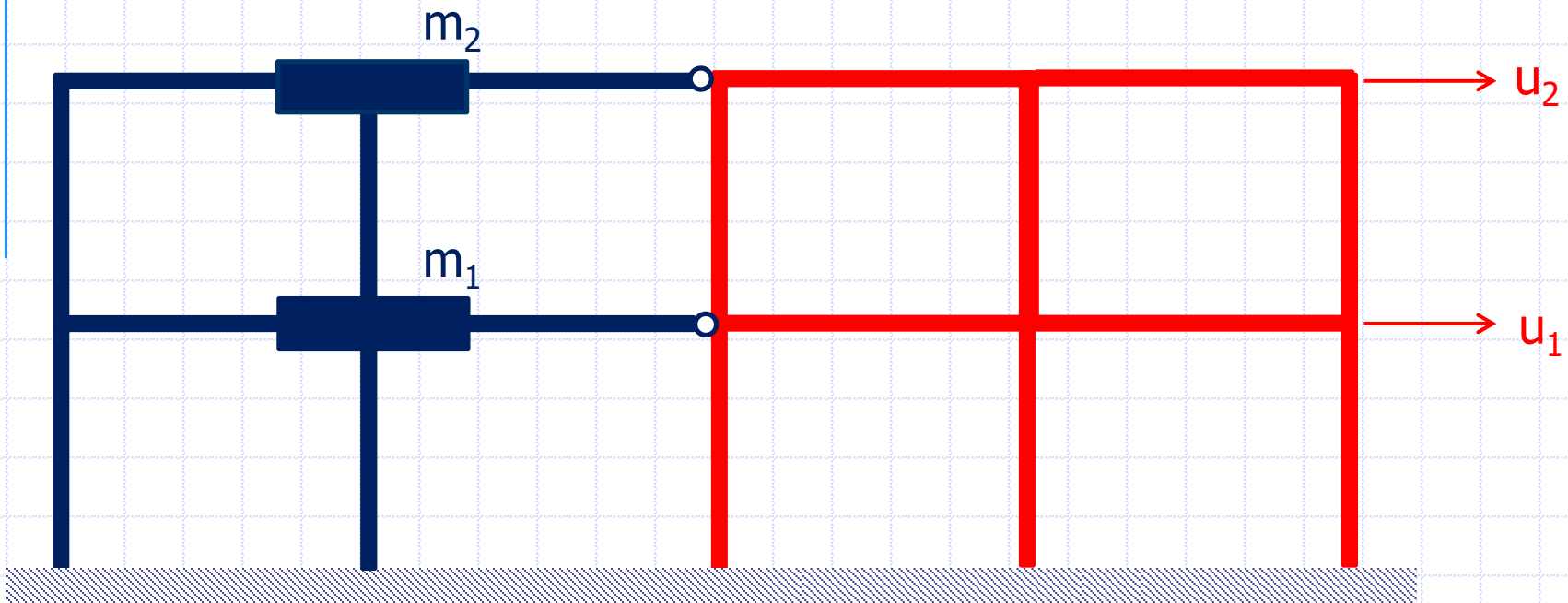
Red : Experimental
Blue: Analytical





Substructuring Cases

CASE 6: MULTI-BAY MULTI-STORY FRAME with ANALYTICAL SUBSTRUCTURING



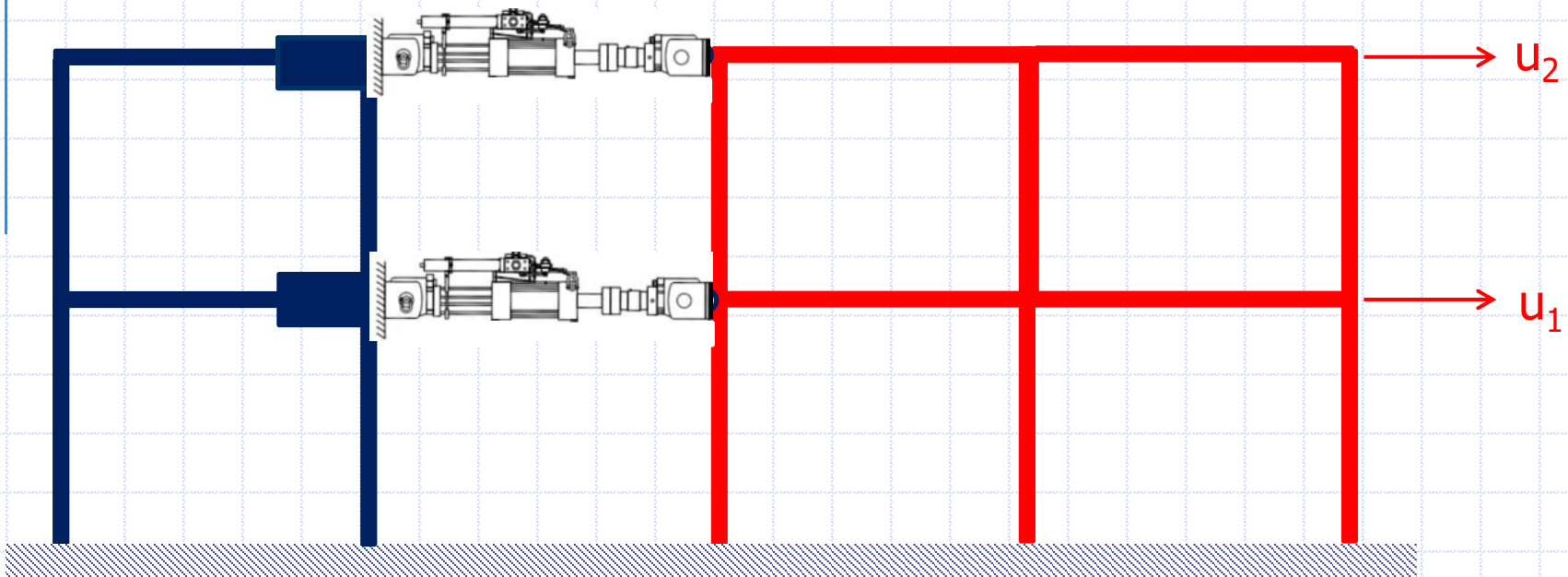
Red : Experimental
Blue: Analytical





Substructuring Cases

CASE 6: MULTI-BAY MULTI-STORY FRAME with ANALYTICAL SUBSTRUCTURING



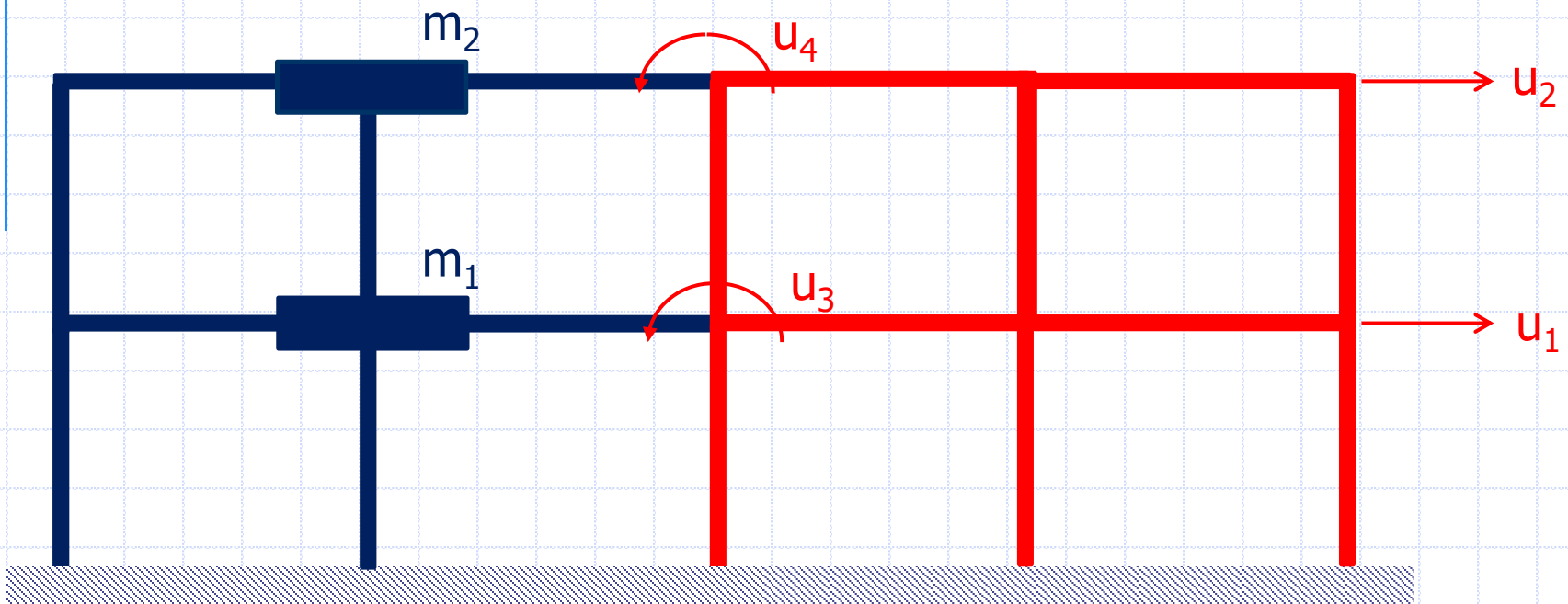
Red : Experimental
Blue: Analytical



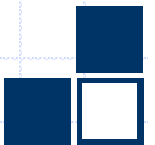


Substructuring Cases

CASE 6-1: MULTI-BAY MULTI-STORY FRAME with ANALYTICAL SUBSTRUCTURING



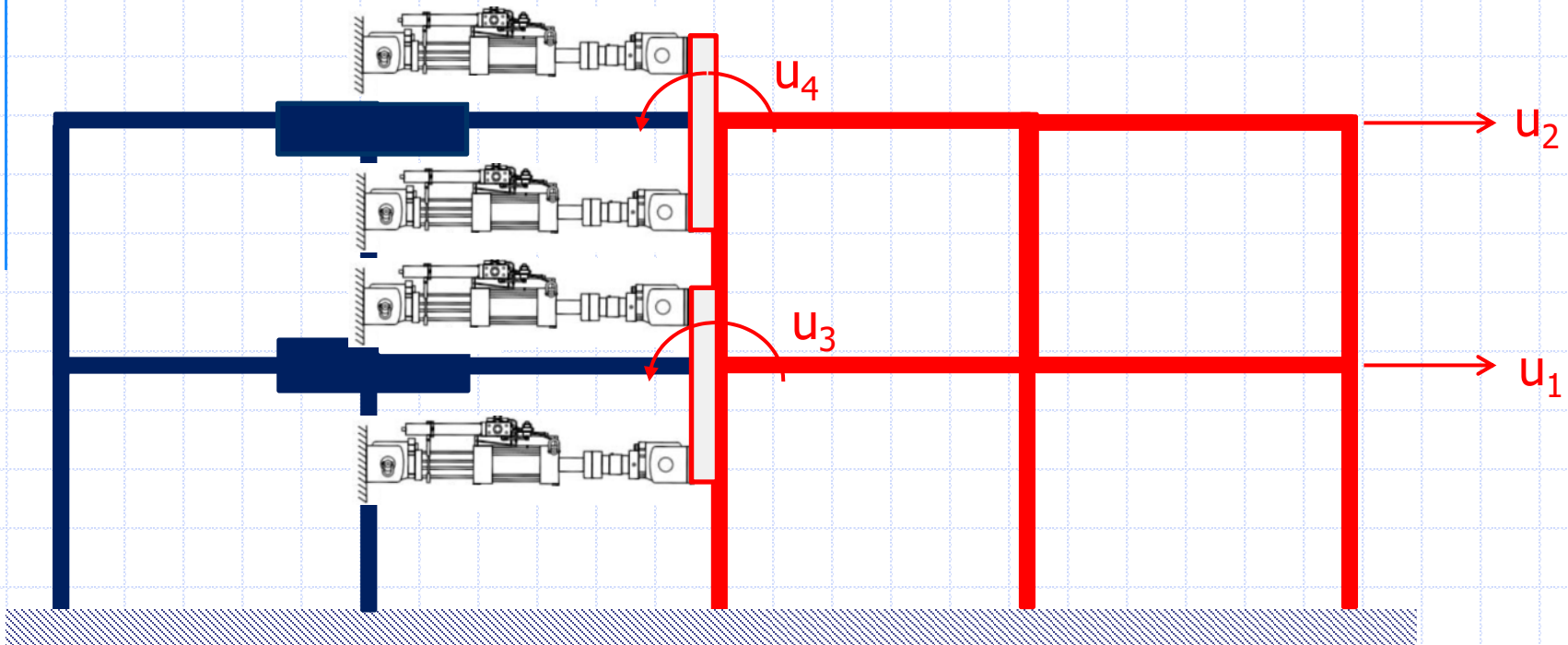
Red : Experimental
Blue: Analytical



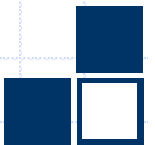


Substructuring Cases

CASE 6-1: MULTI-BAY MULTI-STORY FRAME with ANALYTICAL SUBSTRUCTURING



Red : Experimental
Blue: Analytical





Integration Methods

Analytical Simulation + Experimental Simulation = Hybrid Simulation

All the integration methods developed for **analytical simulations** are not suitable for hybrid simulation

Example: The most common and standard integration method for **analytical simulation**,
Implicit Newmark Integration



Integration Methods

Implicit Newmark Integration

$$\mathbf{p}_{i+1} - m\ddot{\mathbf{u}}_{i+1} - c\dot{\mathbf{u}}_{i+1} - \mathbf{p}_r(\mathbf{u}_{i+1}) = \mathbf{0}$$

Equilibrium equation

$$\ddot{\mathbf{u}}_{i+1} = \frac{1}{(\Delta t)^2 \beta} (\mathbf{u}_{i+1} - \mathbf{u}_i) - \frac{1}{\Delta t \beta} \dot{\mathbf{u}}_i - \left(\frac{1}{2\beta} - 1 \right) \ddot{\mathbf{u}}_i$$

$$\dot{\mathbf{u}}_{i+1} = \frac{\gamma}{\Delta t \beta} (\mathbf{u}_{i+1} - \mathbf{u}_i) - \left(\frac{\gamma}{\beta} - 1 \right) \dot{\mathbf{u}}_i - \Delta t \left(\frac{\gamma}{2\beta} - 1 \right) \ddot{\mathbf{u}}_i$$

Difference equations

Equilibrium and difference equations represent a nonlinear system of equations, $f(\mathbf{u}_{i+1}) = \mathbf{p}_{i+1} - m\ddot{\mathbf{u}}_{i+1} - c\dot{\mathbf{u}}_{i+1} - \mathbf{p}_r(\mathbf{u}_{i+1}) = \mathbf{0}$

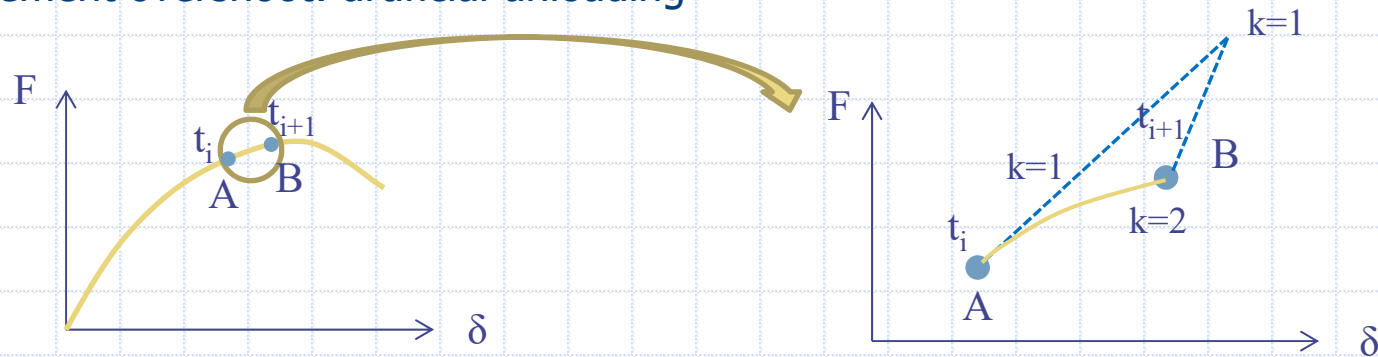
which can be solved using **iterative methods** such as Newton-Raphson method $f'(\mathbf{u}_{i+1}^k) \Delta \mathbf{u}_{i+1}^k = -f(\mathbf{u}_{i+1}^k)$



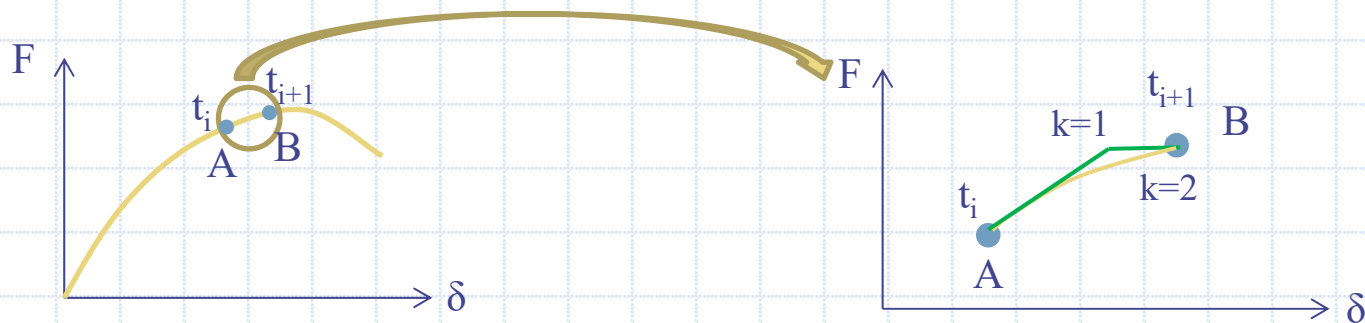
Integration Methods

Iterations of Implicit Newmark are **not suitable** for hybrid simulation:

- Iterations may not converge
- Displacement overshoot: artificial unloading



- Nonuniform displacement increments: velocity and acceleration oscillations within the step





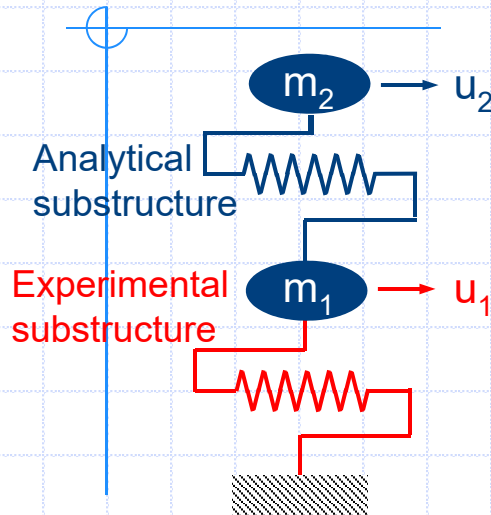
Integration Methods

HS compatible alternative integrators

- Explicit Newmark Integration
 - Operator Splitting Method
 - Implicit Newmark Integration with **Fixed Number of Iterations**
- Do not require iterations**



Simulation Errors



$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_1 \\ \ddot{u}_2 \end{bmatrix} + \begin{bmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{bmatrix} \begin{bmatrix} \dot{u}_1 \\ \dot{u}_2 \end{bmatrix} + \begin{bmatrix} -f_a + f_e \\ f_a \end{bmatrix} = - \begin{bmatrix} m_1 \\ m_2 \end{bmatrix} \ddot{u}_g$$

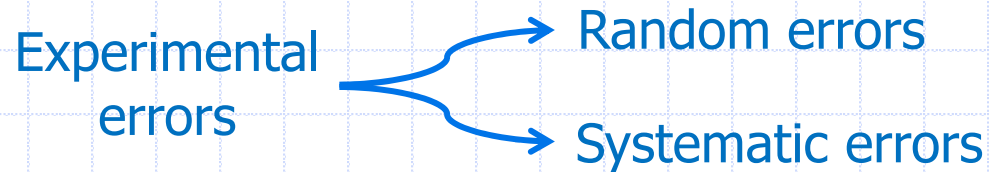
1. Apply $u_{1,i+1}$ to the test specimen
2. Measure the corresponding force $f_{e,i+1}$

- ❑ Reliability of a hybrid simulation depends on the accuracy of f_e
- ❑ All the errors that occur during stages 1 and 2 are experimental errors and affect hybrid simulation





Simulation Errors



Random errors:

- They have no distinguishable pattern and generally no specific physical effects can be anticipated.
- **Examples:**
 1. Random electrical noise in wires and electronic systems
 2. Random rounding-off or truncation in the A/D conversion of electrical signals
- They do not introduce significant errors to hybrid simulation.



Simulation Errors

Experimental systematic errors:

- They may lead to error propagation and numerical instability
- **Examples:**
 1. Measurement errors
 2. Hybrid simulation technique (ramp and hold, continuous, real-time)
 3. Servo-hydraulic closed control loop

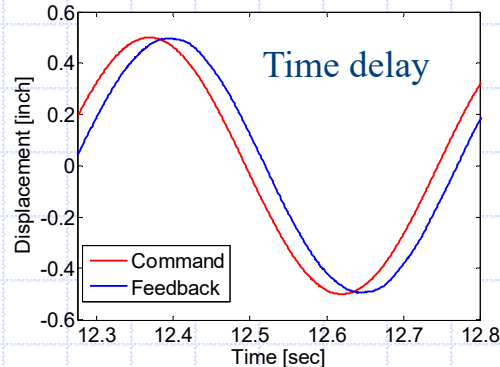
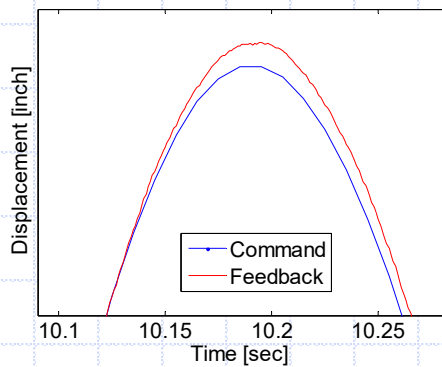
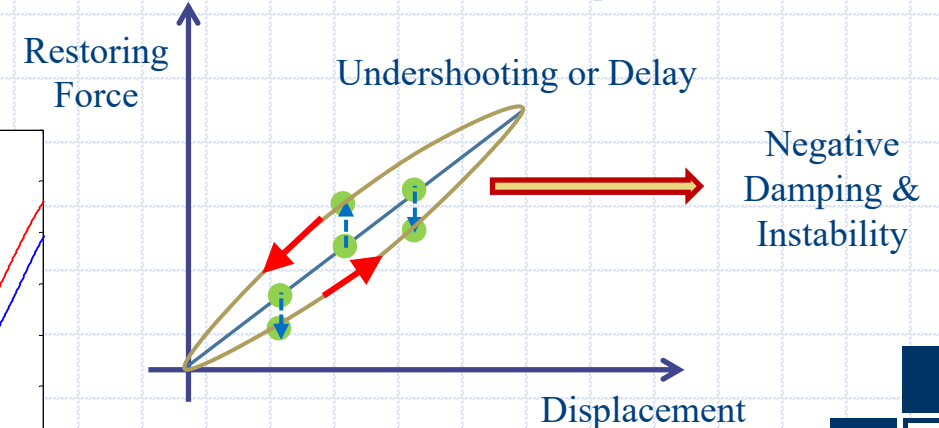
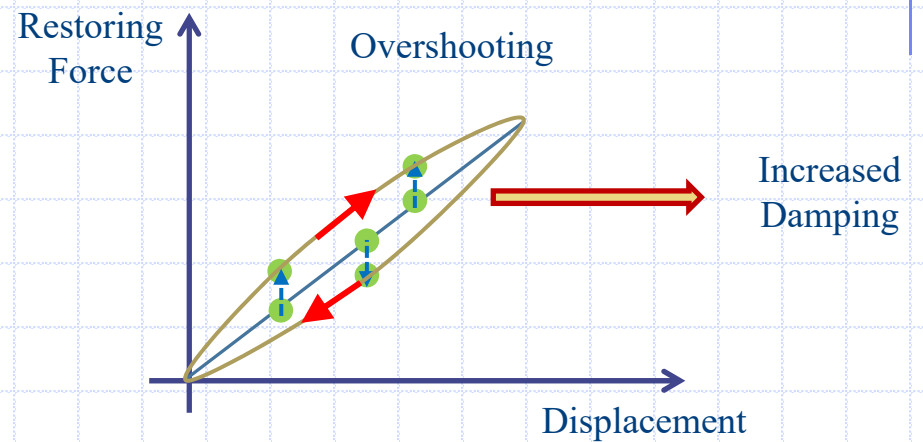
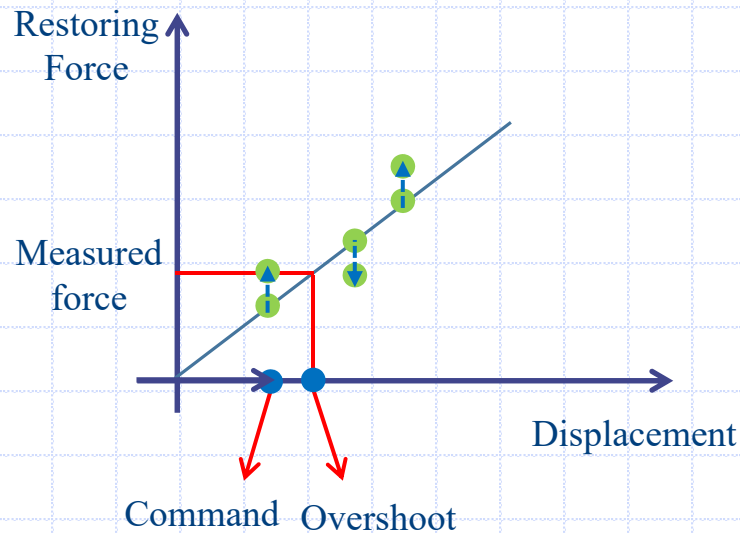
1. Measurement errors

- ❑ Errors in load cells & displacement transducers of actuators due to:
 - a. Calibration
 - b. Friction or slop in the attachments
 - c. A/D and D/A conversions



Simulation Errors

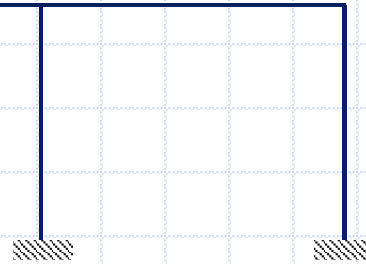
Control-loop errors (Errors in displacement tracking): Demonstration of the effect of control-loop errors



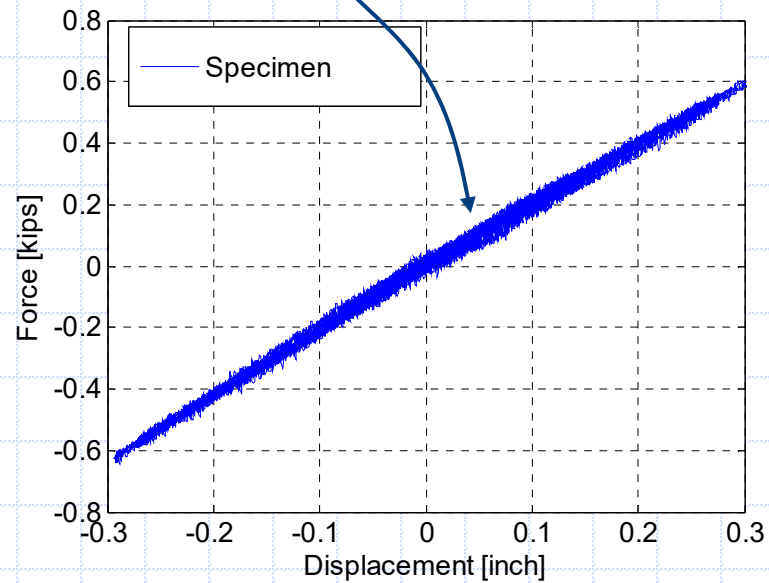


Simulation Errors

Control-loop errors: Demonstration tests



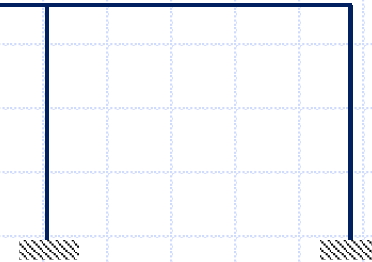
$T = 0.5 \text{ sec}$
 $\zeta = 5\%$



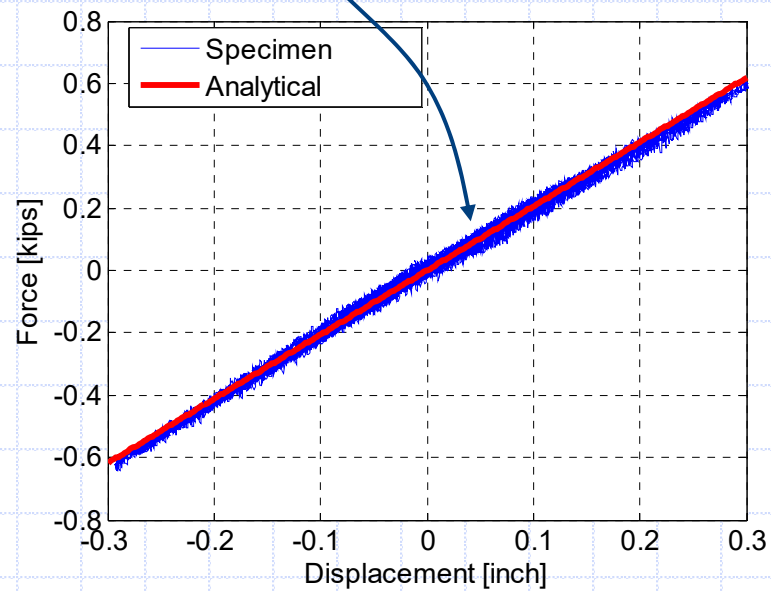


Simulation Errors

Control-loop errors: Demonstration tests



$T = 0.5 \text{ sec}$
 $\zeta = 5\%$

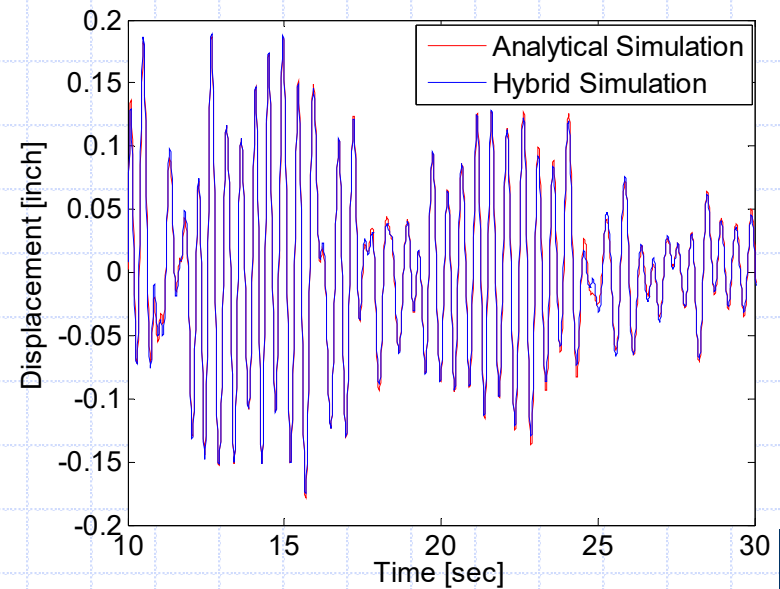
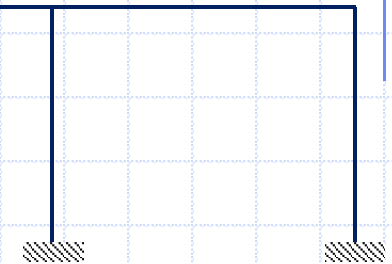
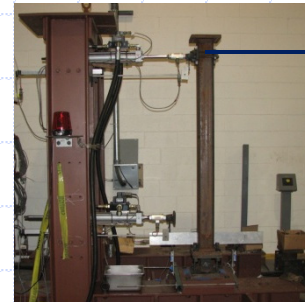
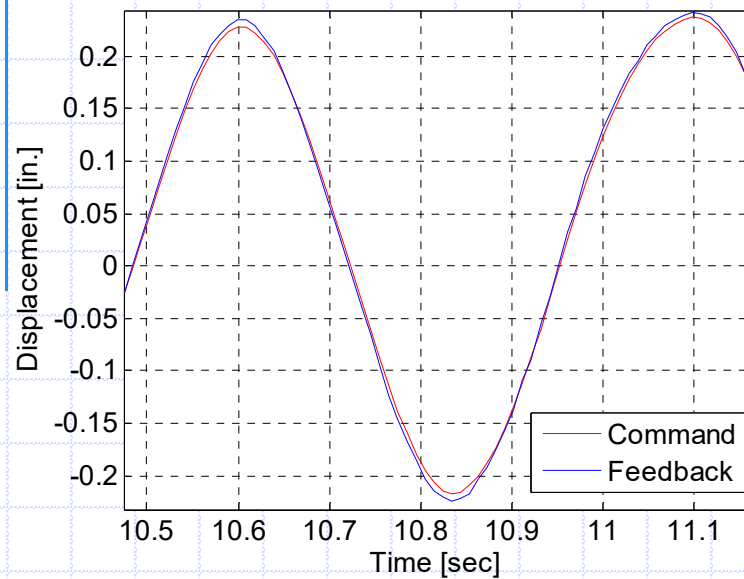




Simulation Errors

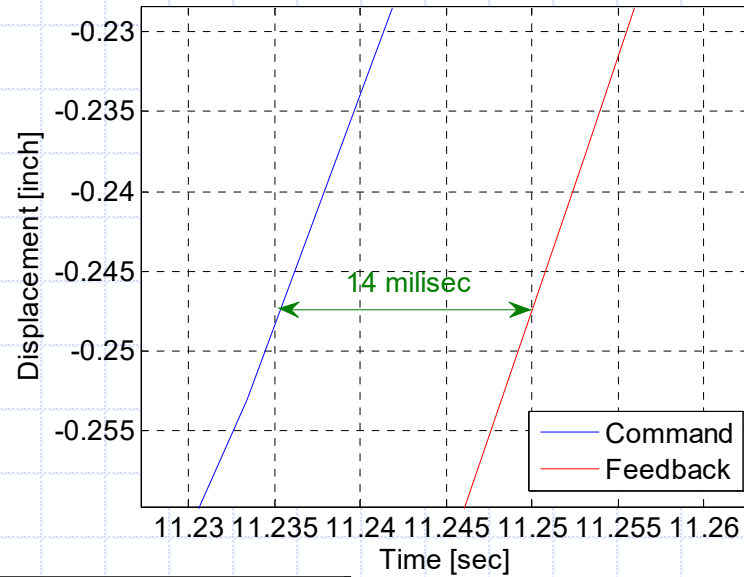
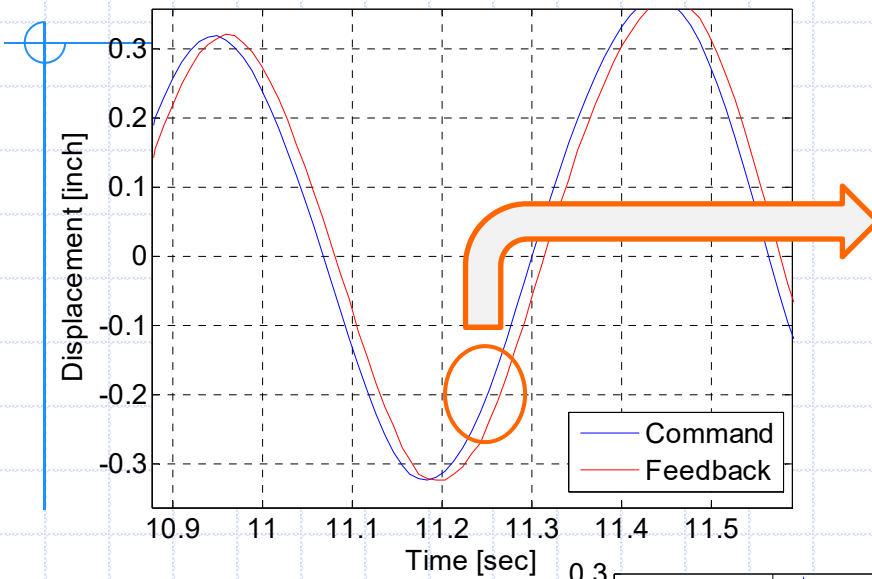
Control-loop errors: Demonstration tests

No time delay

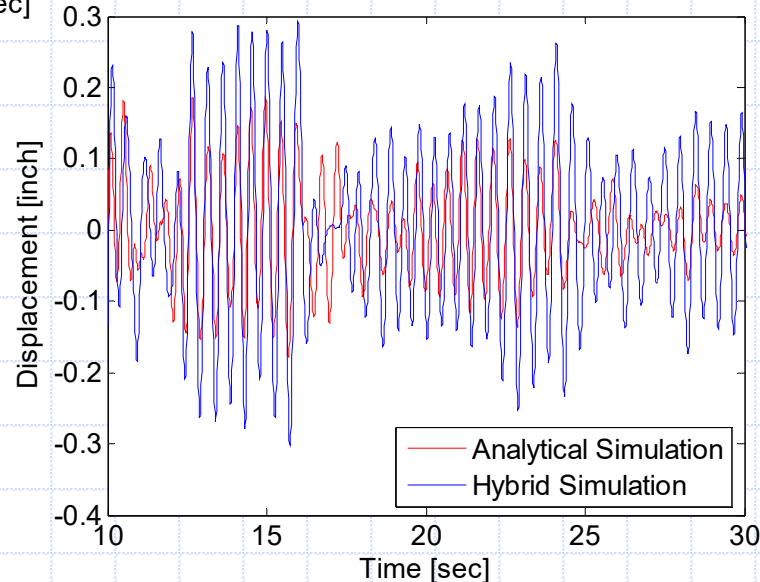




Simulation Errors



14 msec time delay
introduced artificially
by adjusting the
feed-forward gain

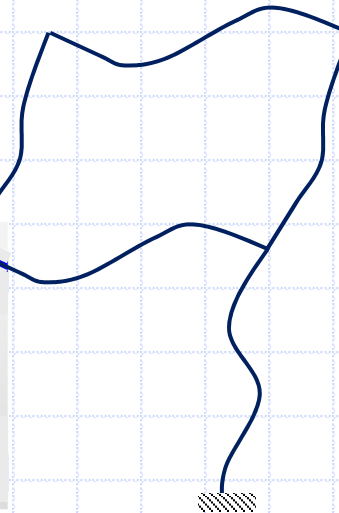




Simulation Errors

Control loop errors: Error identification using free vibration

Step 1: Push the hybrid structure, generally in the first mode, to a displacement within the linear range

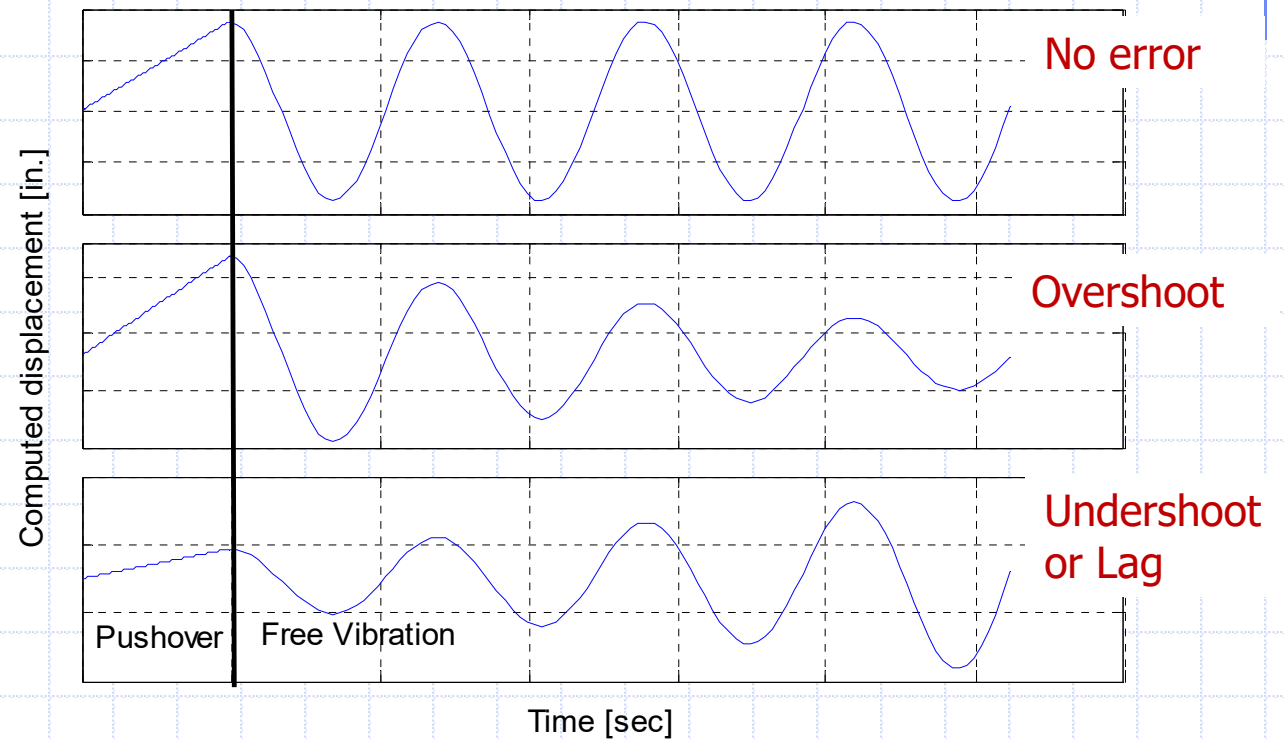
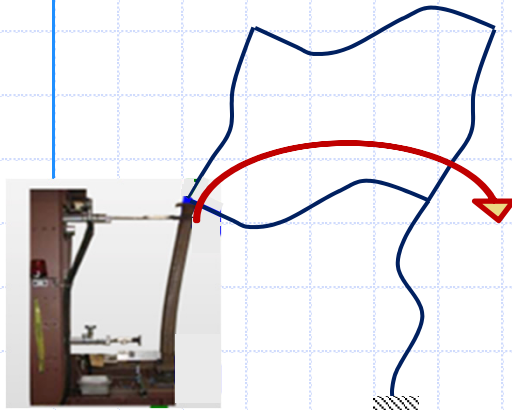


Step 2: Run the free vibration hybrid simulation test from this displaced configuration



Simulation Errors

Control loop errors: Error identification using free vibration





Simulation Errors

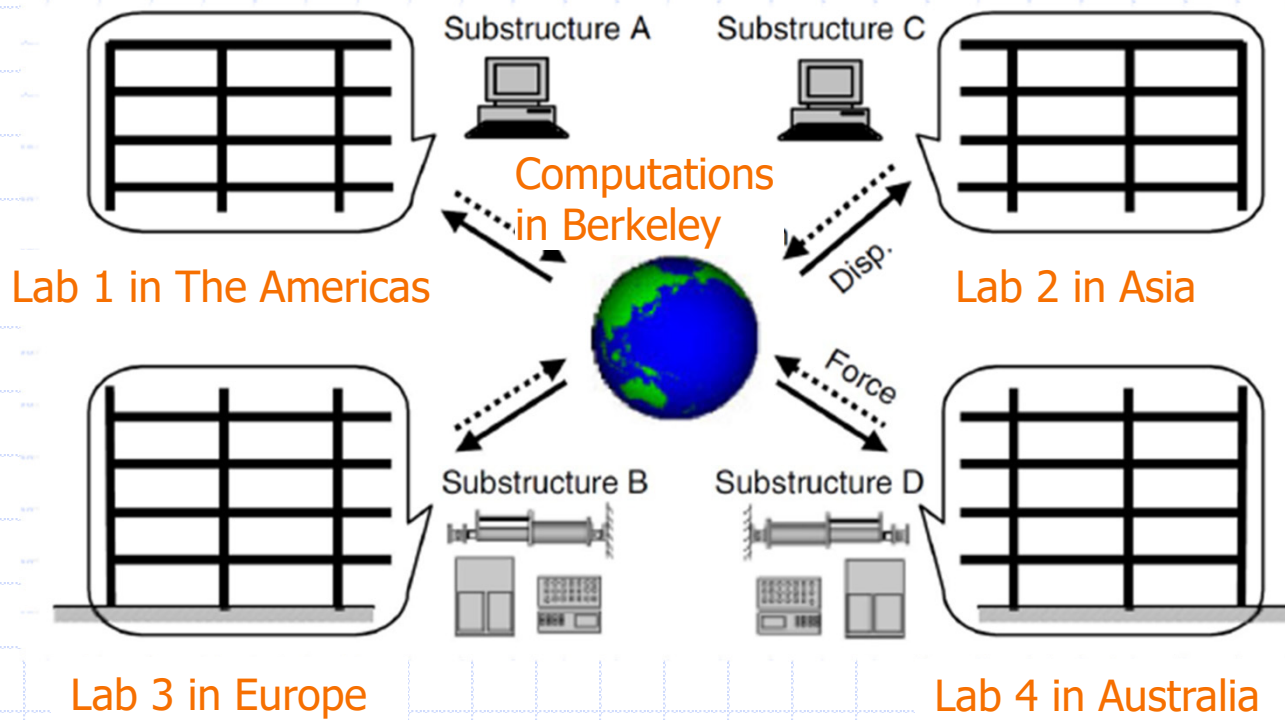
Methods to Reduce the Effects of Errors

- Error Compensation Methods
- Integration Methods with Numerical Damping
- Tuning
- Advanced Control Methods



HS Related Research

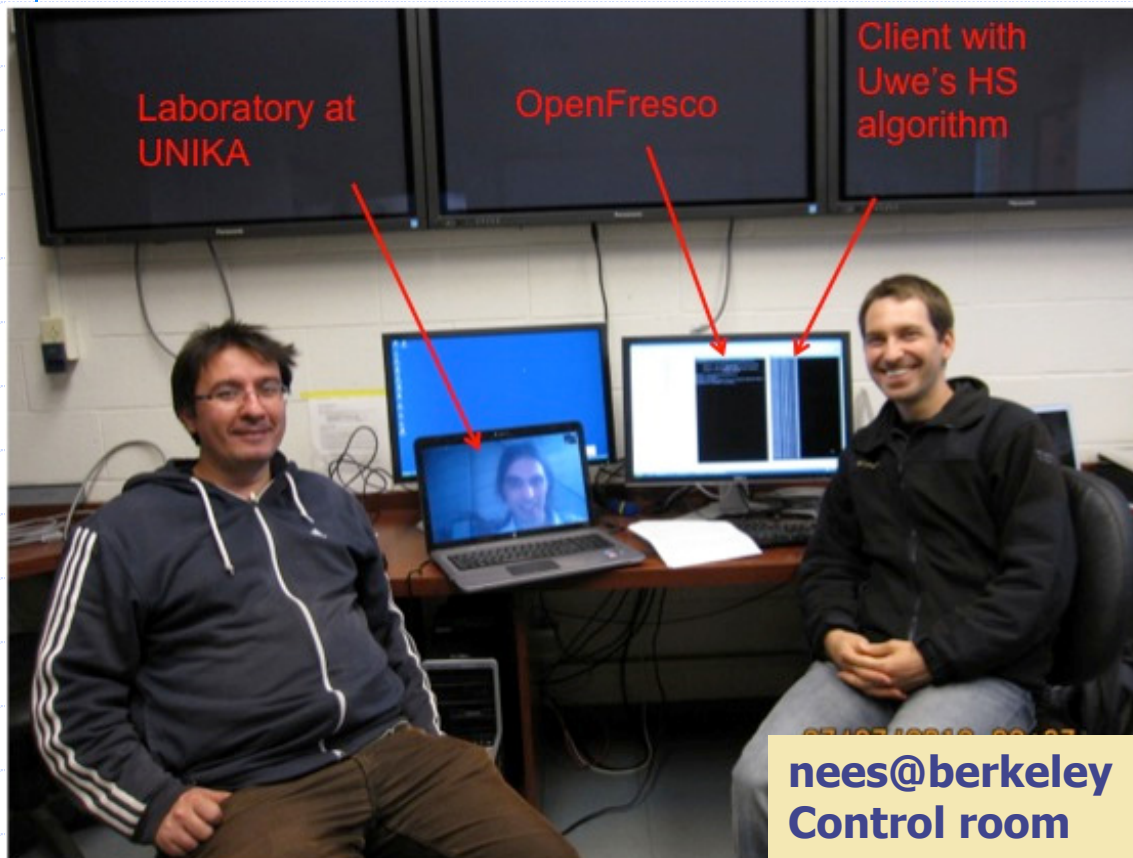
Geographically Distributed HS





Geographically Distributed HS

Geographically distributed HS test between
nees@berkeley and UNIKA, Germany



Experimental substructure:

Friction device and a fixed tuned-mass-damper @UNIKA

Analytical substructure:

SDOF mass with viscous damping @Berkeley

OpenFresco: The Open-source Framework for Experimental Setup and Control

<http://openfresco.berkeley.edu/>



Real-time Hybrid Simulation (RTHS)

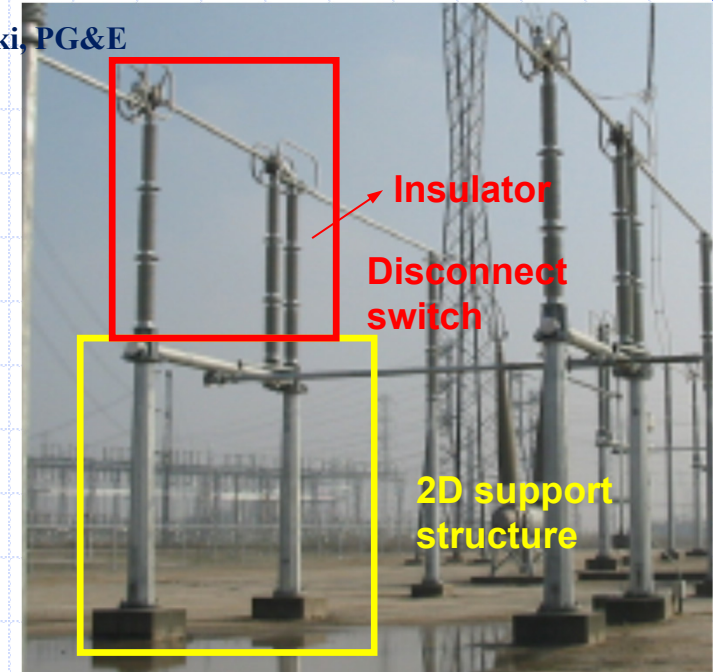
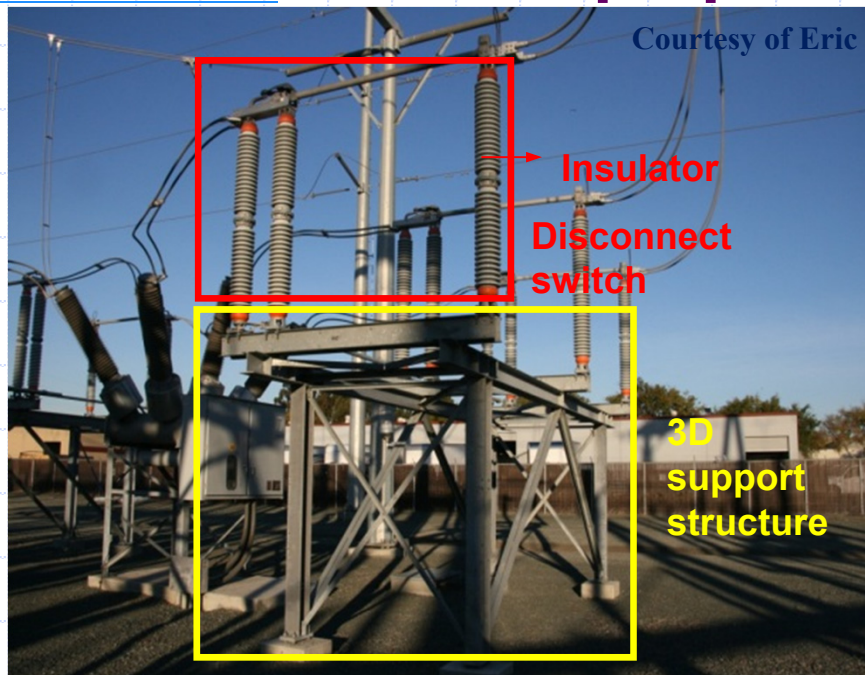
➤ Requirement for real time:

Loading rate = Computed velocity

- **Slow HS:** Sufficient for most cases when rate effects are not important.
- **RTHS:** Essential for rate-dependent materials and devices, e.g. viscous dampers, friction pendulum isolators or polymer insulators.



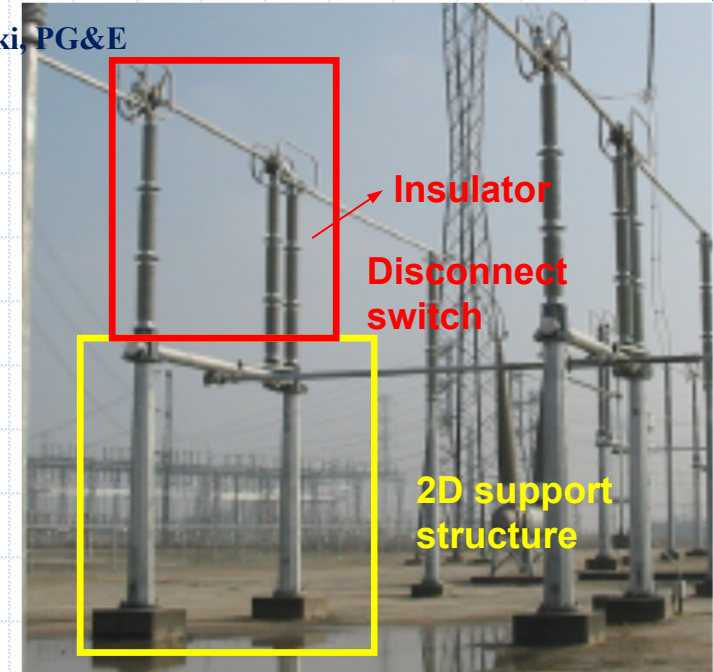
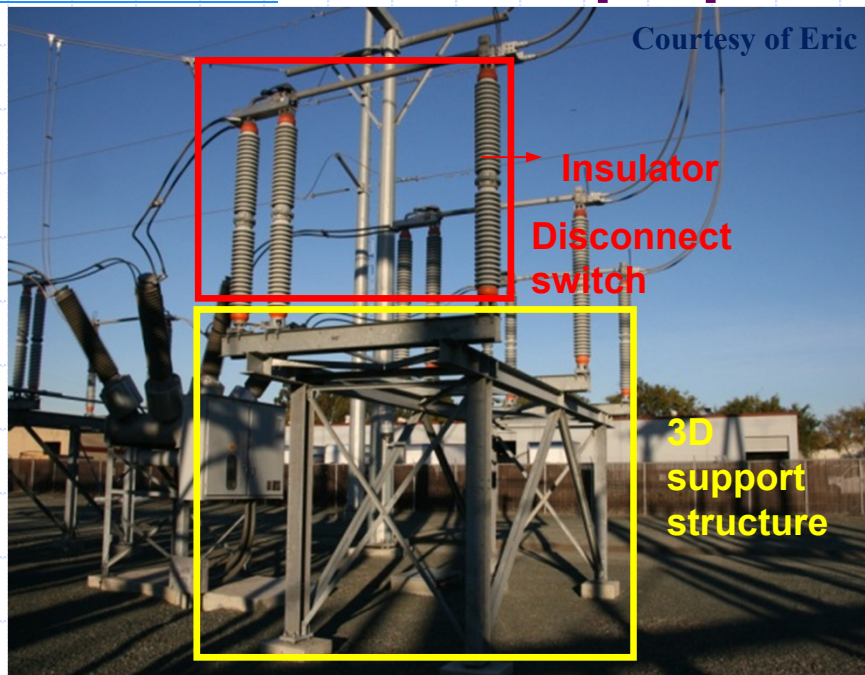
Use of HS for Testing of Electrical Equipment



- ❑ Electrical equipment in substations are typically mounted on support structures to provide sufficient clearance of the ground, and to integrate them into the design of the substation.
- ❑ Support structures are generally **steel frames with well defined geometry and material properties**. Therefore they are suitable to be modeled in the computer as **analytical substructure**.
- ❑ Electrical equipment generally have **complex geometry and material properties with larger uncertainty**.
- ❑ HS provides an **effective, efficient and economic** testing opportunity by combining the electrical equipment testing with support structure modeling.



Use of HS for Testing of Electrical Equipment



- ❑ HS provides an **effective, efficient and economic** testing opportunity by testing of the electrical equipment and modeling of the support structures.
1. Application I: Evaluation of the Effect of Support Structure Stiffness and Damping on Porcelain and Polymer Insulators
 2. Application II: Full Disconnect Switch Tests in Open and Closed Configurations
 3. Application III: Testing of Interconnected Equipment

Thank you!



nees@berkeley

The George E. Brown, Jr. Network for Earthquake Engineering Simulation

